

Low-Swirl Combustion - An Ultra-Low Emissions Technology for Industrial Heating & Gas Turbines, and Its Potential for Hydrogen Turbines

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Acknowledgements

- **Sponsors**

- ▶ DOE-FE, Office of Clean Power Systems
- ▶ DOE-OE, Distributed Energy Resources
- ▶ California Energy Commission

- **Collaborators**

- ▶ D. Littlejohn D. Yegian, G. Hubbard, K. Hom & I. Shepherd (LBNL)
- ▶ J. Rafter & C. Taylor (Maxon)
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- **Previous Sponsors**

- ▶ DOE-Basic Energy Sciences, Chemical Sciences
- ▶ DOE-Basic Energy Sciences, Laboratory Technology Research
- ▶ California Institute of Energy Efficiency/SoCalGas
- ▶ DOE-EERE, Industrial Technology Programs

What's Unique About Low-swirl Combustion?



- **LSC is a simple, yet sophisticated way to burn gaseous fuels (hydrocarbons & hydrogen) efficiently with very low NO_x emissions by a lower cost and durable burner**

Synopsis

- **Low-swirl combustion is a novel flame stabilization concept that is radically different than conventional approaches**
 - ▶ Basic operating principle derived from experimental observations & analyses
 - ▶ Active research topic not covered by combustion text books
- **LSC is an ultra-low emissions enabling technology for heating and power systems**
 - ▶ Great attributes:
 - **Scalable** - 7 kW (1") to 14 MW (16") burners due to linear characteristics
 - **Robust** - < 2 ppm NO_x (@15%O₂) without exhaust gas clean-up or tight controls, high turndown (30:1), fuel flexible, and reliable
 - **Low-cost** - simple design made of conventional materials, size & form compatible with current designs, and adaptable without re-configuration
- **Analytical tools available to guide future developments**
 - ▶ Less uncertainties & more predictabilities to
 - scale to the capacity and high T, P conditions of large utility turbines
 - develop fuel-flexible low-swirl combustors for gaseous hydrocarbons, landfill gases, biomass gases, refinery gases, syngases and hydrogen

Agenda

Time	Topic
10:05 AM	Introduction to Low-swirl Combustion Theoretical background, review of LSC operating principle, development history, and overview of recent accomplishments.
10:30 AM	Q&A on LSC Technology.
10:35 AM	Technology Transfer Experiences Adaptation and scale-up for Maxon industrial burners Development for Solar Turbine 7 MW Taurus engine
10:55 AM	Q&A on Data from applications
11:00 AM	Empirical model for LSC adaptation to large scale turbines for natural gas, syngas, and hydrogen fuels.
11:15 AM	Q&A on Potential Applications
11:20 AM	Scale-up issues and challenges for IGCC
11:30 AM	Q&A on Scale-up Issues and Challenges
11:35 AM	Needed R&D to address potential Issues and challenges
11:50 AM	Q&A on Needed R&D
11:55 AM	Wrap-up
Noon	Adjourn

Part 1:

Introduction to Low-swirl Combustion

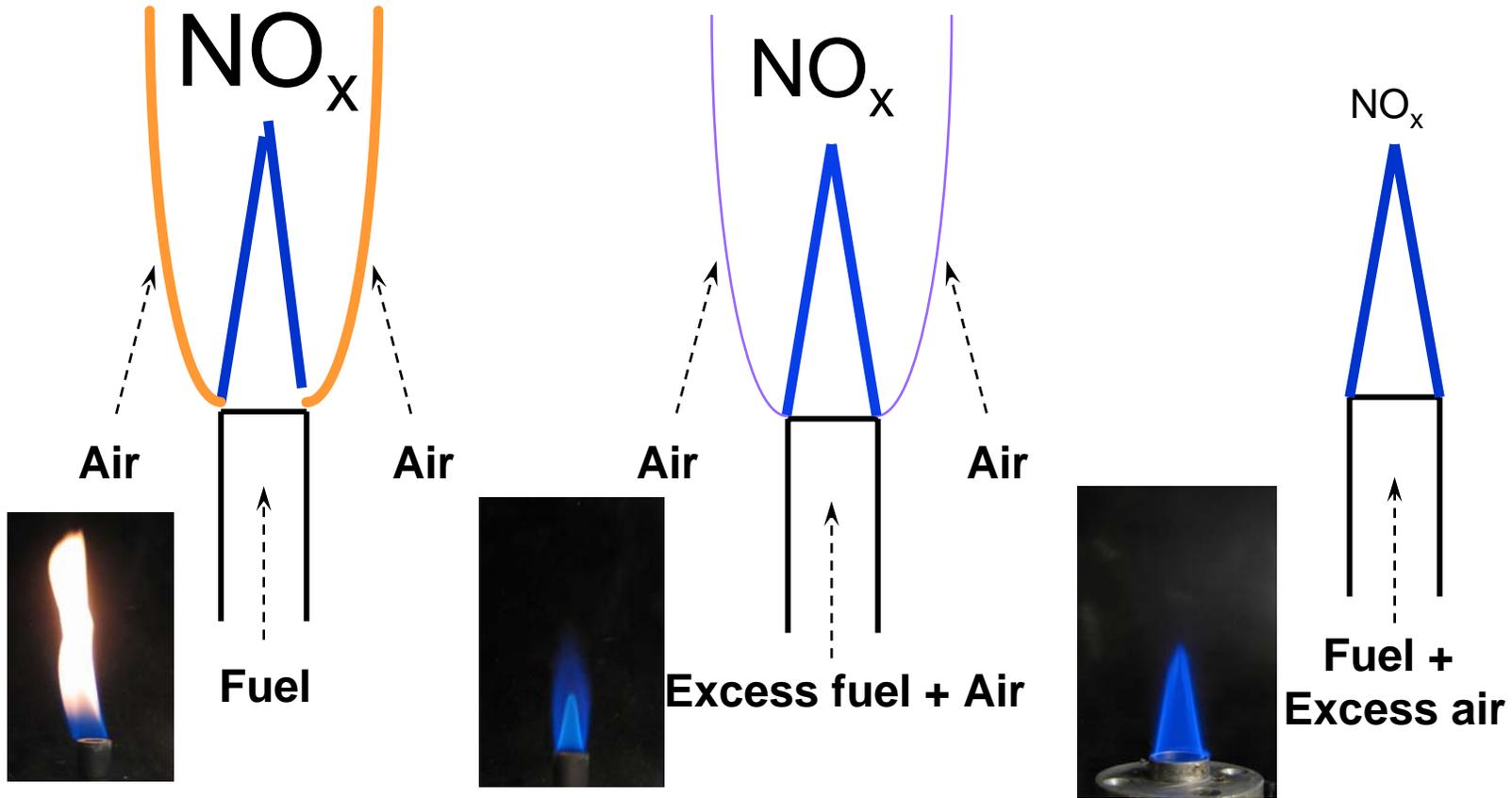
**Theoretical Background,
Review of LSC Principle
Development History**

Overview of Recent Accomplishments

Combustion Science and Technology

- **One of the oldest technologies of mankind**
 - ▶ Created and destroyed civilizations
 - ▶ Combustion provides 83% of US energy
- **Highly refined art of combustion engineering**
 - ▶ Contemporary systems with high efficiency and reliability
 - ▶ Development and testing cycle absorbed much resources
- **Relatively new scientific discipline**
 - ▶ Sophisticated research tools – advent of lasers and computers in the 1970s
 - ▶ Diverse topics and scope - rocket engines, aircraft engines, gas turbines, burners, boilers, incinerators, fires and smoldering combustion
 - ▶ Multi-disciplinary - chemistry, physics, mechanical engineering, chemical engineering, civil engineering, and computational science
 - ▶ Highly specialized practitioners – expertise for each specific combustion system (transportation, power, manufacturing, aerospace, military, fire safety etc.)

Modes of Gaseous Combustion



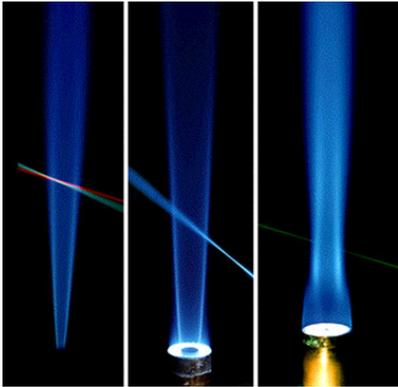
Diffusion Flame
controlled by mixing

Partially Premixed Flame
two reaction zones

Lean Premixed Flame
wave-like flame front

Turbulent Combustion Occurs Naturally in All Practical Systems

- Important research topic of physical sciences
- No unified theory due to differences in combustion modes

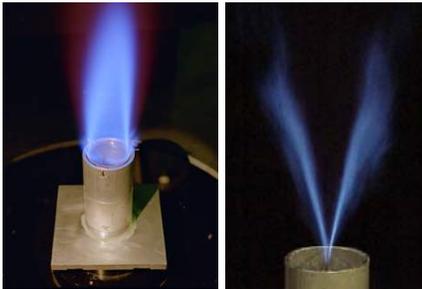


- **Non-premixed (diffusion) flames**

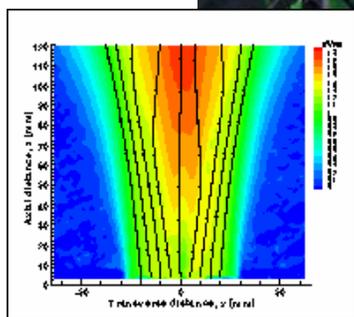
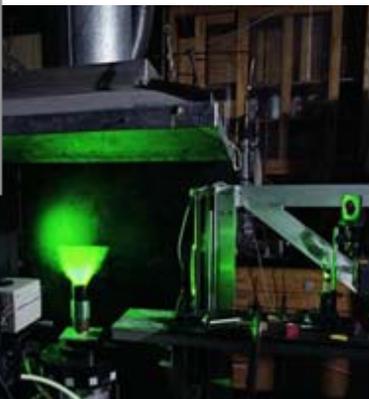
- ▶ Turbulent and molecular mixing control combustion rates, efficiency and pollutant formation
 - Reactions occur at stoichiometric contours passive to turbulence
- ▶ Reaction rate models expressed in terms of species concentrations

- **Premixed flames**

- ▶ Self propagating flame front separates reactants from products
- ▶ Flame front exhibits wave behavior and generates significant feedback to turbulent field through the pressure field
- ▶ Reaction rate models expressed in terms of flame speed



LBNL Combustion Research Focuses on Premixed Turbulent Flames



- **Motivation:** Turbulence and fluid mechanics dominate flame processes
 - ▶ Chaotic and random fluid motions controlling heat release, emissions, & explosion limits
- **Scientific Needs:** Fundamental understanding of flame/turbulence interactions
 - ▶ Laboratory burners with well characterized turbulence fields that offer broad range of operating conditions are critical tools
 - ▶ Statistical data analysis methods to validate theories and simulations

Numerical simulation of a rod-stabilized v-flame validated by detailed velocity and scalar data

Turbulent Combustion as an Applied Research Problem

- **Flames burning in complex turbulent flowfields**
 - ▶ Flame behavior not always predictable
- **Need scientific foundation to reduce R&D cost**
 - ▶ Moving away from handbooks to computational design tools for advanced combustor designs
 - ▶ Improve performance – turndown, emissions, reliability and durability
 - ▶ Reduce production and maintenance costs - avoid costly materials and catalysts for exhaust gas cleanup
 - ▶ Intelligent deployment of sensors for control
 - Characterize flame behavior during power turndown
 - Understand precursors to instability, flashback and blowoff

Flames in Large Industrial Applications and Power Generation Systems

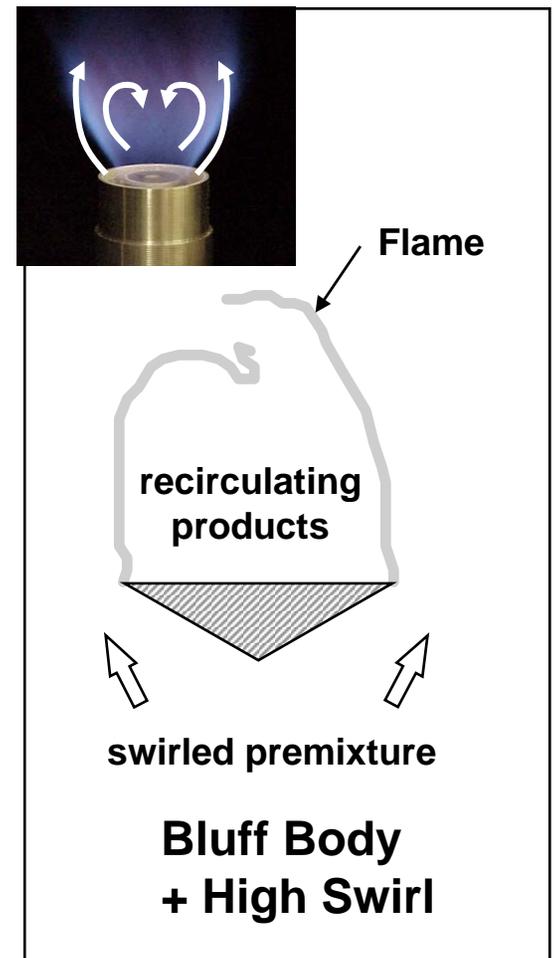
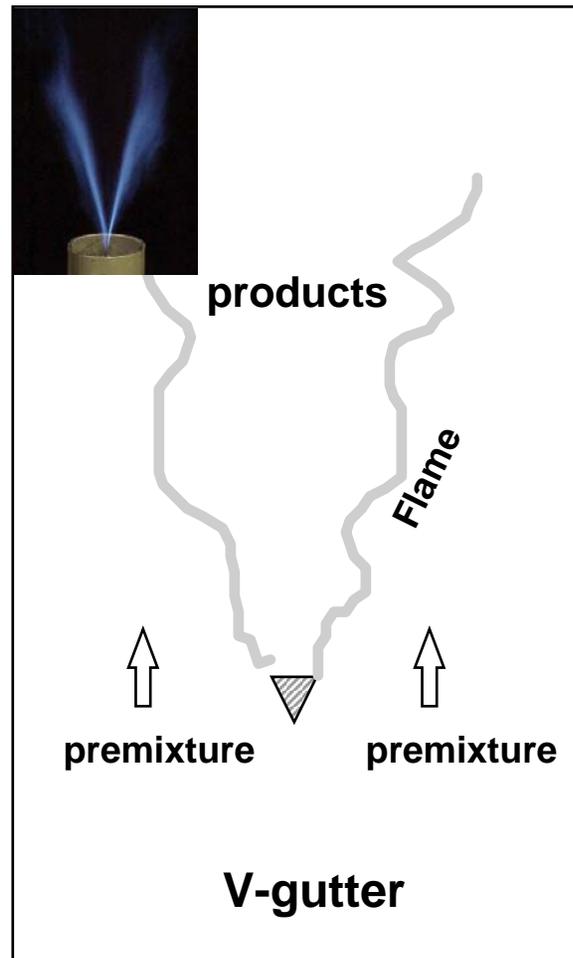
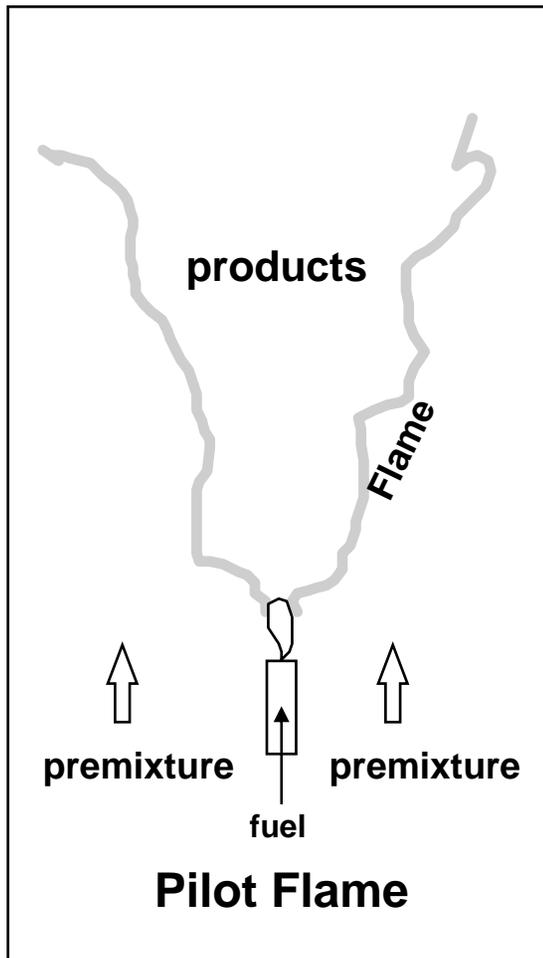
- **Most burners and combustors exceed 99% efficiency**
 - ▶ Design challenges concerned with optimizing efficiency & emissions
- **Non-premixed or partially premixed flames cannot meet stringent emissions rules without costly exhaust gas cleanup**
- **Lean premixed combustion or multi-staged combustion concepts do not require exhaust gas cleanup to meet emissions targets**
- **Tight control and monitoring deemed essential for ultra-low emission**
 - ▶ Navigating startup, load change, and off-normal situations
- **“Stationary sources” subjected to most stringent emission regulations**
 - ▶ Operate continuously with few interruptions (> 20000 hr. non-stop)
 - ▶ Low “Single digit NO_x” requirement planned for most of CA and regions in TX, LA, and Northeastern States
 - ▶ **Near-zero emissions target for DOE-Clear Power Systems**

Issues of Lean Premixed Combustion for Heating and Power

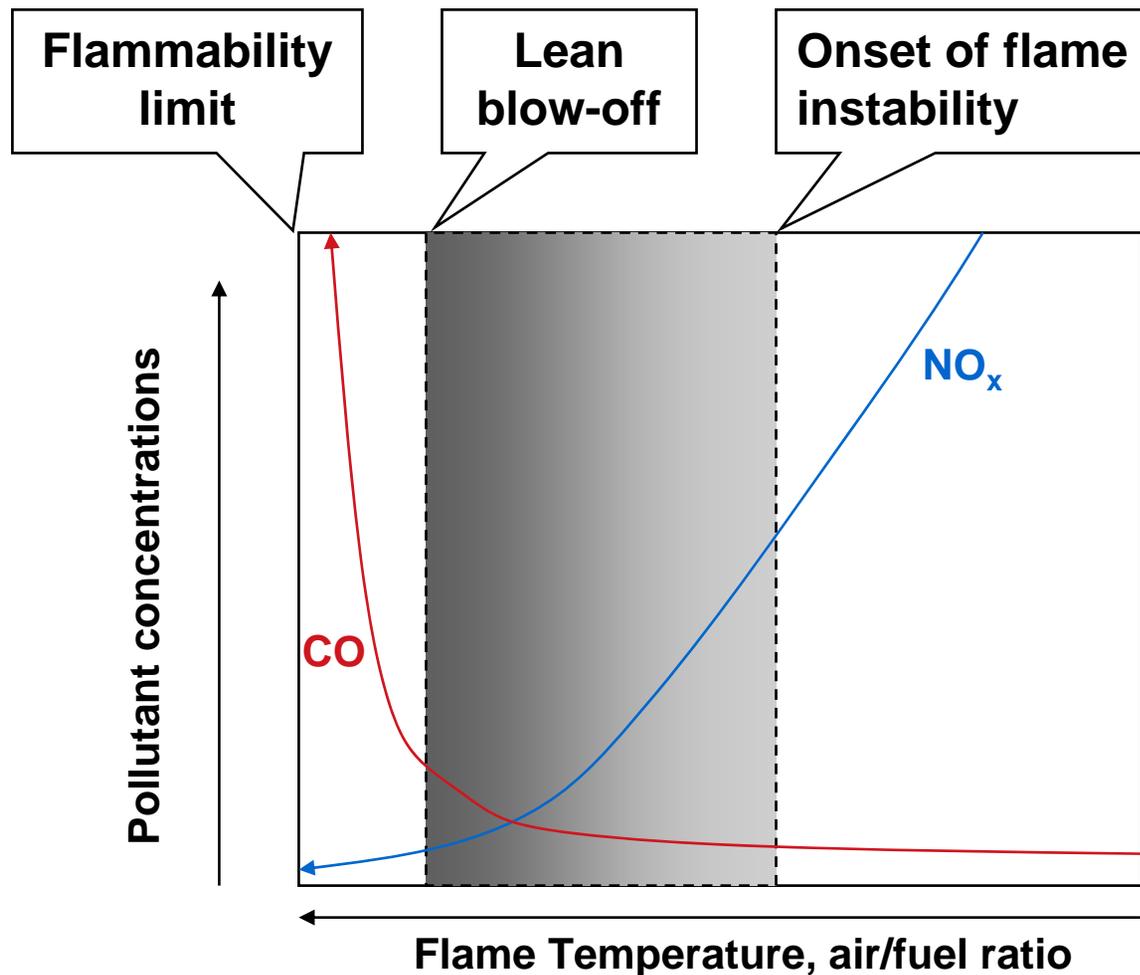
- **Limited knowledge on turbulent flame behavior**
 - ▶ Turbulence, fuel type, operating conditions affect **flame speed**, burning rates, flame size & emissions
- **Flame holders dictate flame shapes and behaviors**
 - ▶ Restriction on operating range (typically 5:1 turn-down vs. 10:1 for non-premixed systems)
 - ▶ Limited fuel flexibility
- **Flame generated flow dynamics**
 - ▶ Strong flame/chamber coupling
 - ▶ Weaker lean flames generates noise and vibrations
 - ▶ flash-back and blow-out hazards

Conventional Premixed Flame Holders

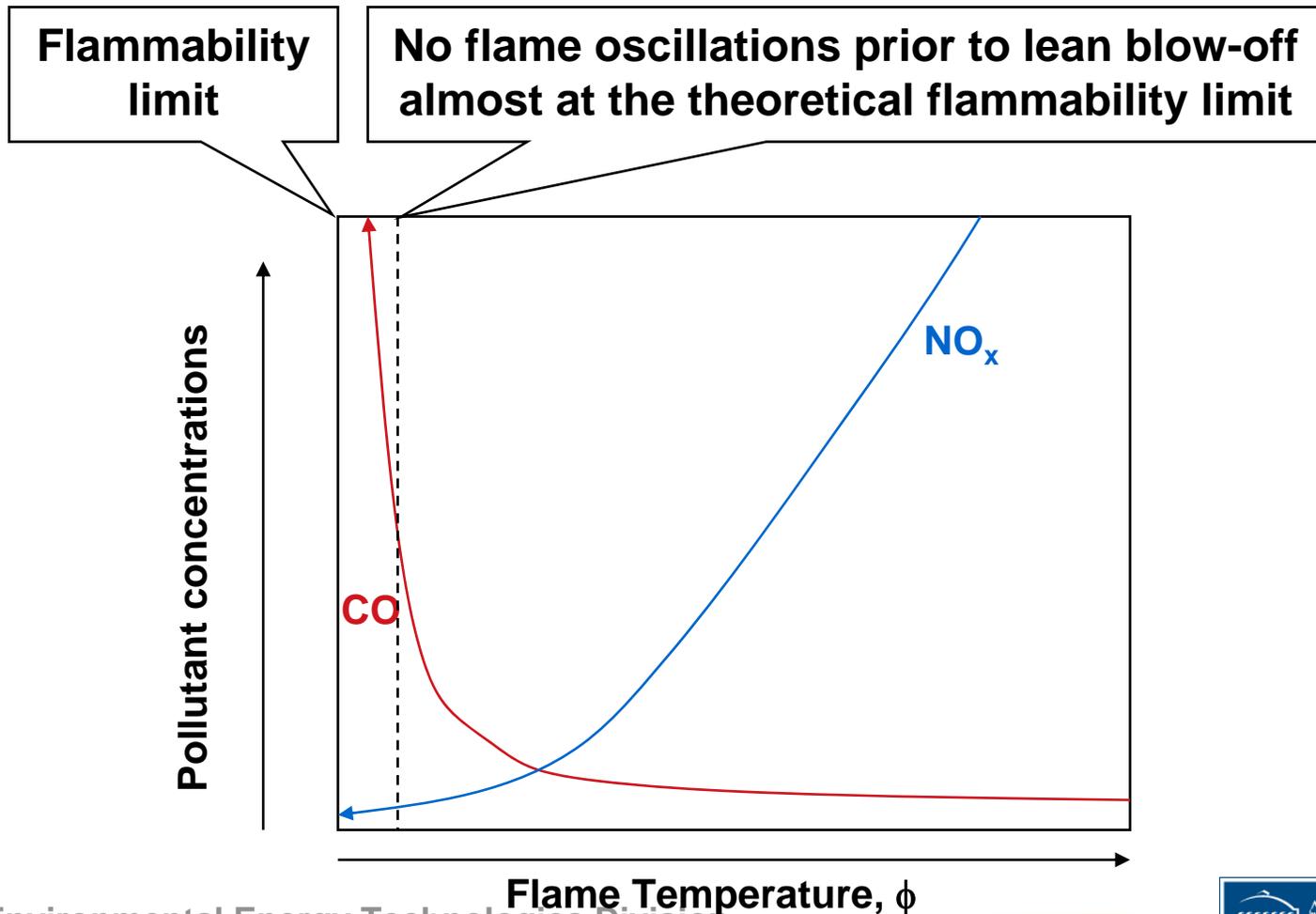
- Theoretical explanation based on continuous ignition provided by pilot flame or in the hot recirculation zone in the wake of the stabilizer
- **“Back flow” within the recirculation zone is considered essential**



Lean Blow-off and Flame Instability Limits of Flame Holders Are Barriers to Reaching Ultra-Low Emissions



Low-swirl Combustion Exploits Aerodynamics to Overcome the Ultra-Low Emissions Barriers



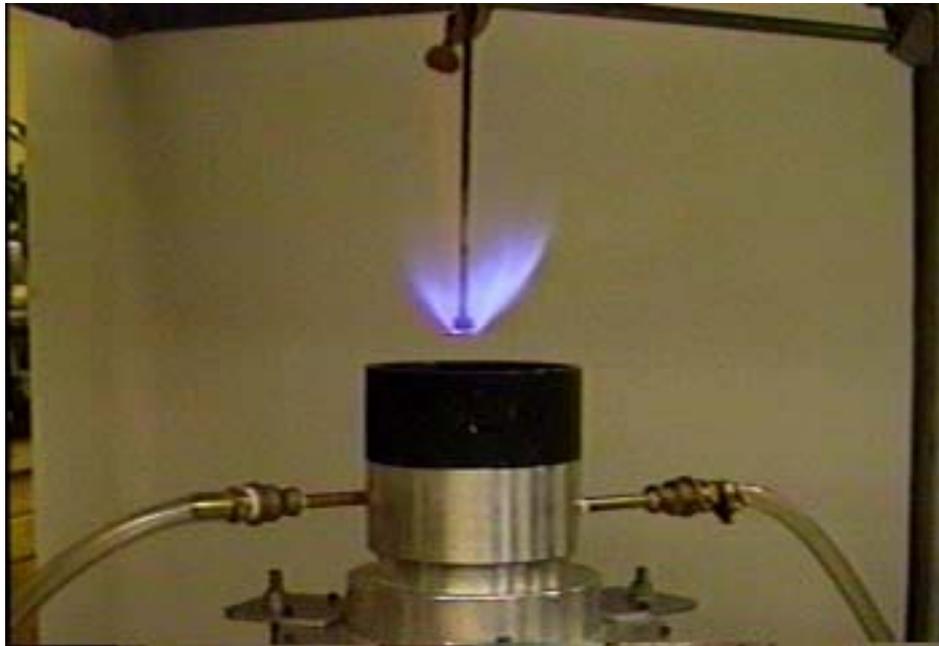
Low-Swirl Combustion (LSC)

- **LSC is a flame stabilization mechanism conceived at LBNL for DOE supported basic research**
 - ▶ **Excellent experimental configuration**
 - Supports stable premixed flames in intense isotropic turbulence and at conditions from near flame-out to high stoichiometries
 - Operating principle derived from experiments and analyses
 - **Exploits turbulent flame speed, S_T** - the most fundamental property of the premixed flames
 - **Not applicable to non-premixed combustion**
 - Adopted by researchers world-wide
 - ▶ **Technology transfer**
 - LSC supports robust lean flames with ultra-low emissions
 - 2 US patents
 - Flame stabilization principle
 - Vane-swirler design
 - Basic knowledge facilitates development of scaling and engineering rules and practical implementation

Video Demonstration of How Low-Swirl Combustion Eliminates the Flame Holder

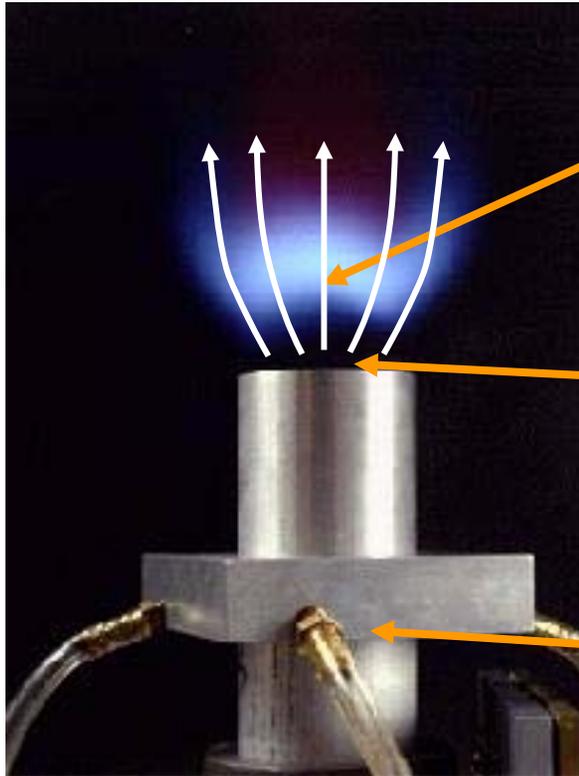
- The first generation low-swirl burners utilize air-jet to generate swirling motion
- The still image below shows the flame anchored to the central stabilizer when the two jets are off

Turning on and gradually increasing the air-jets show transition from anchored to lifted flame



<http://eetd.lbl.gov/aet/combustion/LSC-Info/LSB%20Initiation.mpg>

Low-Swirl Flame Stabilization Exploits Propagating Nature of Premixed Flames



Fuel/Air
mixture

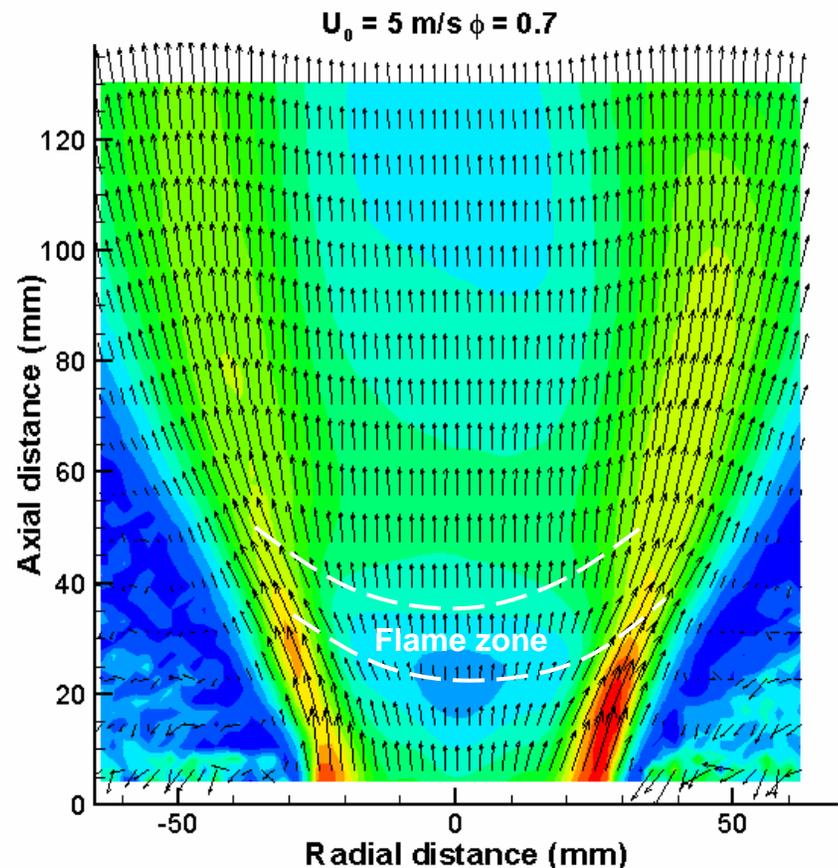
Propagating against the divergent flow, the flame settles where the local velocity equals the flame speed

Flow divergence within the **non-swirling** central region is the key element for flame stabilization

Small air jets swirl the perimeter of the fuel/air mixture but leave the center core flow undisturbed

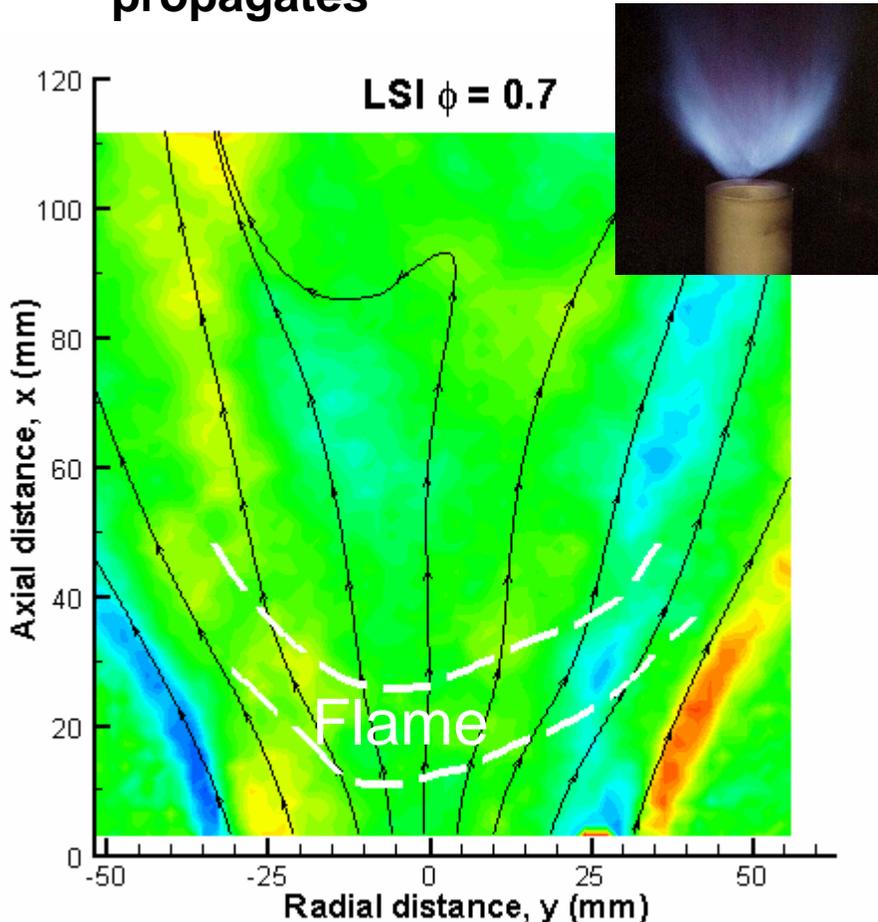
Flame Stabilization Mechanism Characterized by Laser Diagnostics

- LSC does not requires “back flow” to stabilize turbulent premixed flames
- Flow divergence rate controlled by swirl rate
- Turbulence intensity provides the feedback to increase flame speed as flow velocity increases
- Flame position invariant through wide range of inflow velocities and fuel/air ratios due to flowfield similarity
- Flame flow coupling described by top level model to guide engineering design

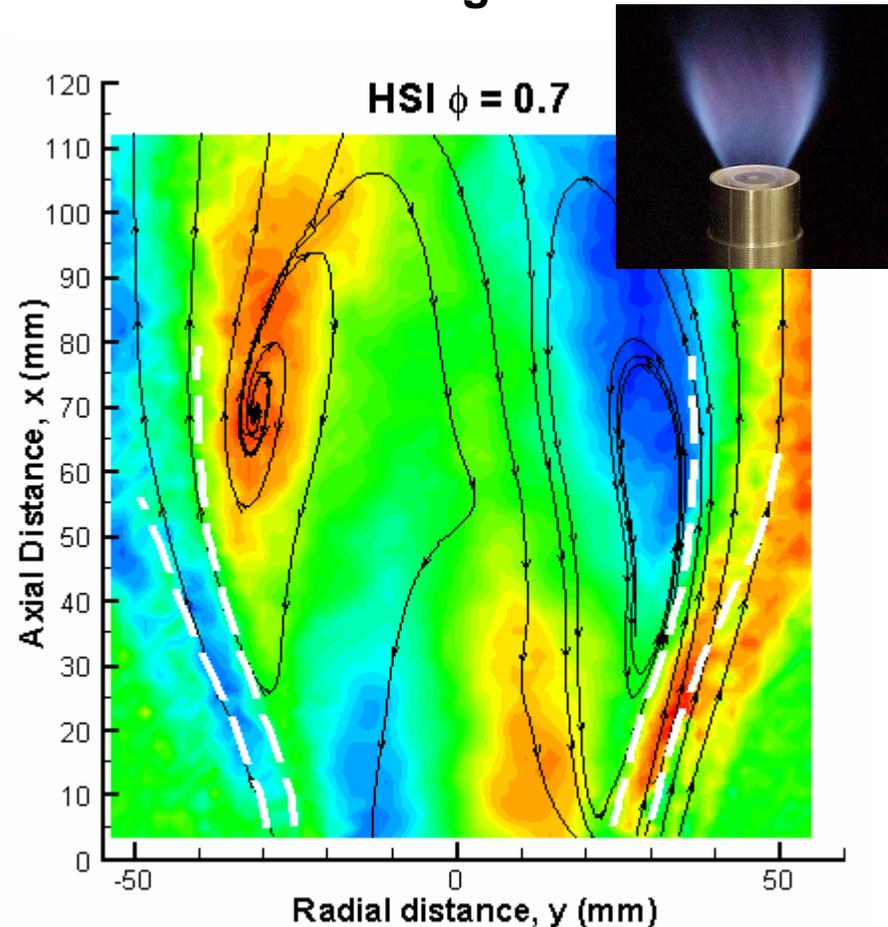


Comparing Flowfields of Low-swirl and High-swirl

- **Low-swirl injector** generates flow divergence where the flame freely propagates



- Conventional **high-swirl injector** generates recirculation flow for flame anchoring



LSC is Fundamentally Different than High-Swirl Flame Stabilization

- **High-swirl combustion processes dictated by fully developed large recirculation zone**
 - ▶ hot combustion products trapped in the vortex **continuously igniting** the surrounding swirling reactants
 - ▶ **nonlinear flame flow coupling** involving vortex breakdown and precessing vortex core
 - ▶ **Lower level models required for engineering design using computational fluid dynamics**
- **Onset of intermittent flame detachments from centerbody are precursors to blowout**
- **Need to mitigate instabilities & oscillations at ultra-lean conditions**
 - ▶ Staged “rich-quench-lean” method
 - ▶ Active feedback control
 - ▶ Catalytic pilot or combustor

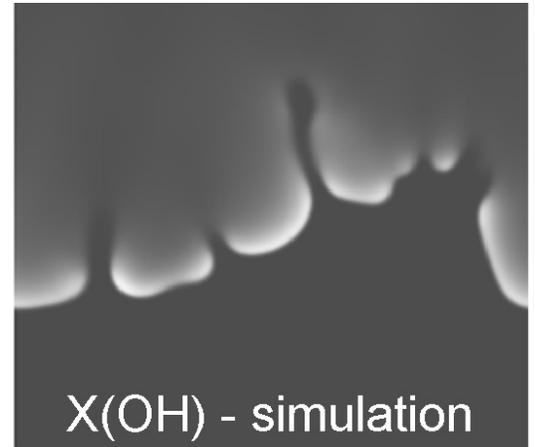
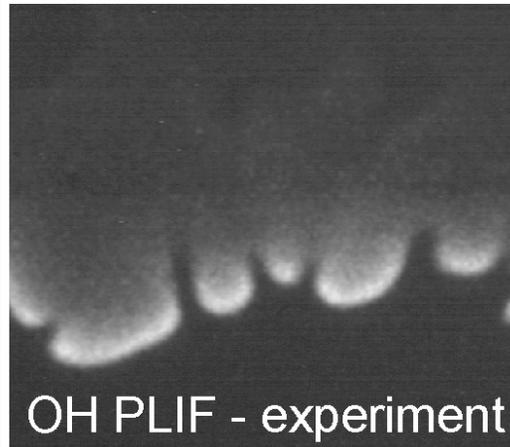
Low Swirl vs. High Swirl for Flame Stabilization

	Low Swirl	High Swirl
Principle	Flame propagates freely at its turbulent flame speed	Vortex traps hot products for continuous ignition of fresh mixtures
Approach	Generation of divergent flow with no recirculation	Generation of tight recirculation zone
Flame/ turbulence interaction	Flame developing in isotropic turbulence with low shear stresses is less prone to fragmentation	Flame developing in high shear region leads to flame fragmentation and occasional detachment
Instability	No distinct characteristic frequency due to absence of recirculation	Characteristic frequencies associated with recirculating vortex

LSC as a Research Tool

- LSC produces a lifted premixed turbulent flames giving free access to laser diagnostics
- Near planar flame brush locally normal to the approach flow gives a close approximation of a slightly stretched 1D premixed turbulent flames for comparison with models
- Developed a benchmark low-swirl burner with Swedish and German scientists for validation of models and simulations

- **Turbulent H₂/air Flames Studied in a Low-swirl Burner**



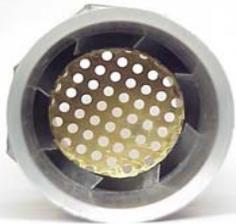
- LBNL computational researcher developing 3D direct numerical simulations of turbulent flames using detailed chemical kinetics
- Measurements and simulation show H₂/air flames are locally quenched and calls question to the validity of conventional flame models for H₂/air flames (31st Sym. Paper)

First Technology Transfer Project Supported by DOE-LTR (1994-1997)

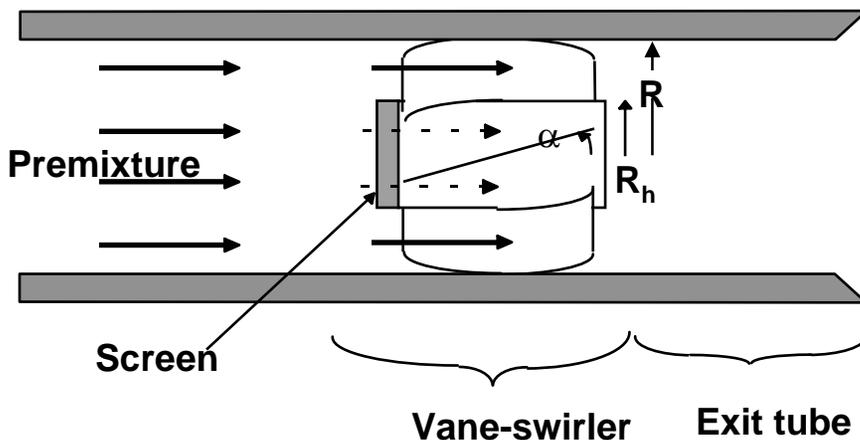
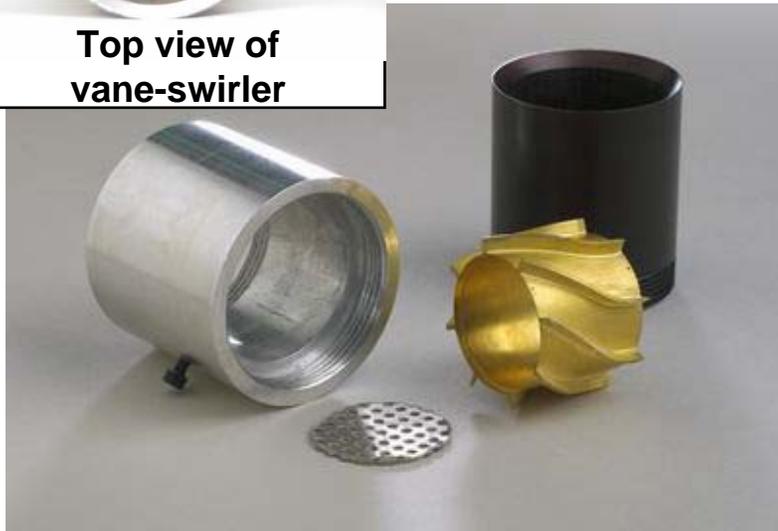


- **Adaptation to pool heaters**
 - ▶ Design and test LSBs that meets the operational requirements of 15 KW to 100 KW units
 - ▶ Sizes similar to laboratory LSBs
 - ▶ Non-modulated systems, no turndown requirement
- **Issues**
 - ▶ **need simpler design requiring only one flow supply (NO JETS)**
 - ▶ firing sideways or downwards to attain > 85% efficiency
 - ▶ stable inside chamber
 - ▶ cannot compromise on energy efficiency
 - ▶ cost must be lower than metal fiber burner (< \$100/per unit)

Vane Swirler for LSC



Top view of vane-swirler



- Reactants pass through two passages – swirled and unswirled
- Non-swirling core inhibit flow recirculation
- Perforated screen covering center channel balances the ratio between swirling and non-swirling portions
- Modified definition of swirl number, S

$$S = \frac{2}{3} \tan \alpha \frac{1 - R^3}{1 - R^2 + [m^2 (1/R^2 - 1)^2] R^2}$$

- Center channel to injector radii ratio, $R = R_c/R_b$
- Vane angle, α
- Flow split between center channel and swirl annulus, m

Vane-Swirler for LSB Can Be Made From Simple and Low-Cost Materials

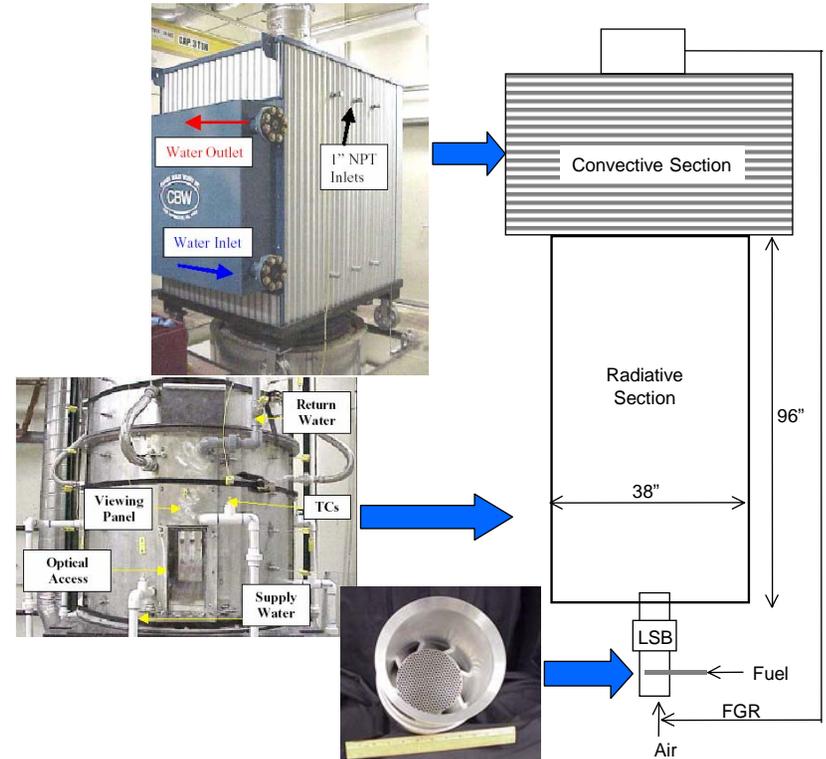


This burner is made of PVC and plastic to showcase the uniqueness of LSB

- Vane-LSB produces the same flame shape as Jet-LSB
- Lifted flame does not transfer significant heat to burner
- Estimated fabrication cost for pool heaters < \$10/unit

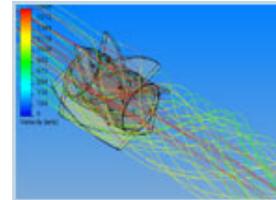
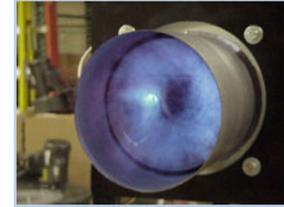
LSC for Industrial Heating

- Scaled to larger capacities
 - ▶ Laboratory studies to determine scaling approach and establish scaling rules
 - ▶ Developed and tested prototypes for large boilers with or without flue gas recirculation
 - ▶ Further development for direct heat applications
- Maxon Corp. licensed LSC in 2002 for ultra-low NO_x burners for industrial heating, baking and drying
 - ▶ < 9 ppm NO_x (@ 3% O₂) guaranteed
 - ▶ “Achieved industry best emissions without sacrificing cost or performance”

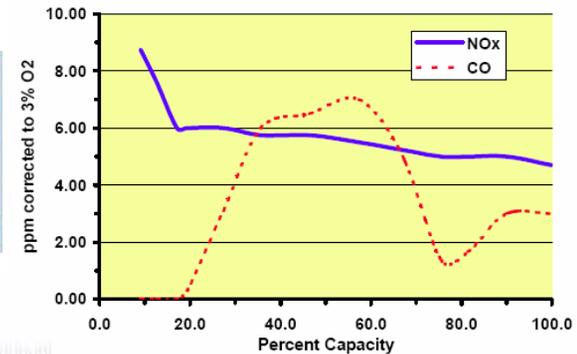


Maxon Has Two LSB Product Lines

- 4-7 ppm NO_x (@3%O₂)
= 1.5-2.5 ppm NO_x (@15%O₂)
- M-PAKT burners (0.5 – 3.5 MMBtu/hr) available since 9/03
 - ▶ 2", 4" and 6" burner diameter
 - ▶ Fuel flexible with natural gas, propane and butane
 - ▶ 10:1 turndown without pilot assistance
 - ▶ Hundred of units installed
 - ▶ Improve product quality (paint curing & food processing)
 - ▶ 1st unit operating continuously since 2/02
- OPTIMA SLS gas/liquid dual-fuel burners (12 - 50 MMBtu/hr) introduced in 2006
 - ▶ 8", 10, 12" and 16" burner diameters
 - ▶ enhanced 13:1 turndown
 - ▶ backup liquid fuel firing
 - ▶ Two prototypes installed & several units in production



Typical Emissions



Development for Gas Turbines

- **DOE Office of Electricity - adaptation of LSC to Solar Turbines' Taurus 70 engine that utilize natural gas, renewable fuels and on-site generation fuels**
 - ▶ Proved LSC concept for gas turbines
- **California Energy Commission - development of a packaged combined heat and power system consisting of a boiler with a Coen Micro-NO_x burner and a Elliott microturbine with a LSC combustor**
- **DOE Fossil Energy - new project to develop low-swirl combustion as an enabling technology for advanced turbines in IGCC power plants utilizing high H₂ fuels**

Part 2

Technology Transfer Experiences

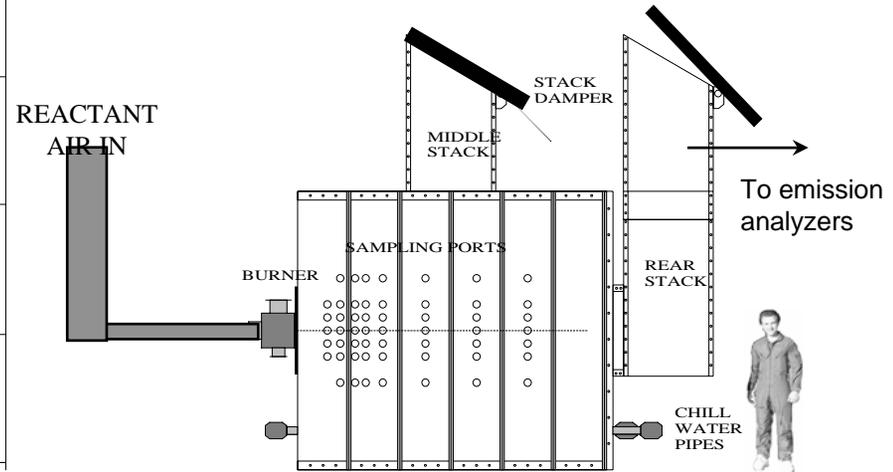
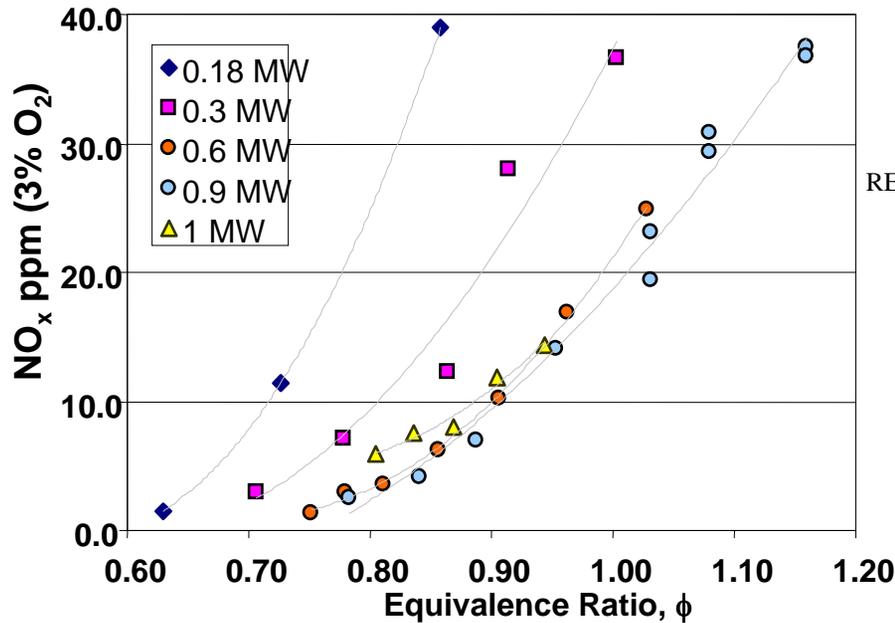
**Adaptation and scale-up for
Maxon industrial burners**

**Development of low-swirl injectors
for Solar Turbine Taurus 70 engine**

Adaptation of LSC to Industrial Burners

- DOE-EERE Industrial Technologies Program fostering “*Scientific approach for ‘smart’ adaptation to a broad range of process heat and boiler applications*”
 - ▶ Targeted 300 KBtu/hr to 50 MMBtu/hr burners
- *Established scaling rules*
 - ▶ Grounded on low-swirl combustion principle
 - ▶ Considered blowout and flashback characteristics
 - ▶ Evaluated trade-off/benefit between two scaling approaches
 - Higher flow velocity vs larger burner diameter
 - ▶ Optimized burner to combustion chamber size and geometry

Laboratory Experiments to Gain Scaling Information



- Comparing LSBs of different sizes (2 – 5") in furnace and boiler simulators with and without FGR
 - ▶ Vane shape, screen placement and small changes in swirl number have little effect on flame noise, flame stability, & lean blow off
 - ▶ NO_x emissions depend primarily on air/fuel ratio (i.e. flame temp.)
 - ▶ Observed 30:1 turndown

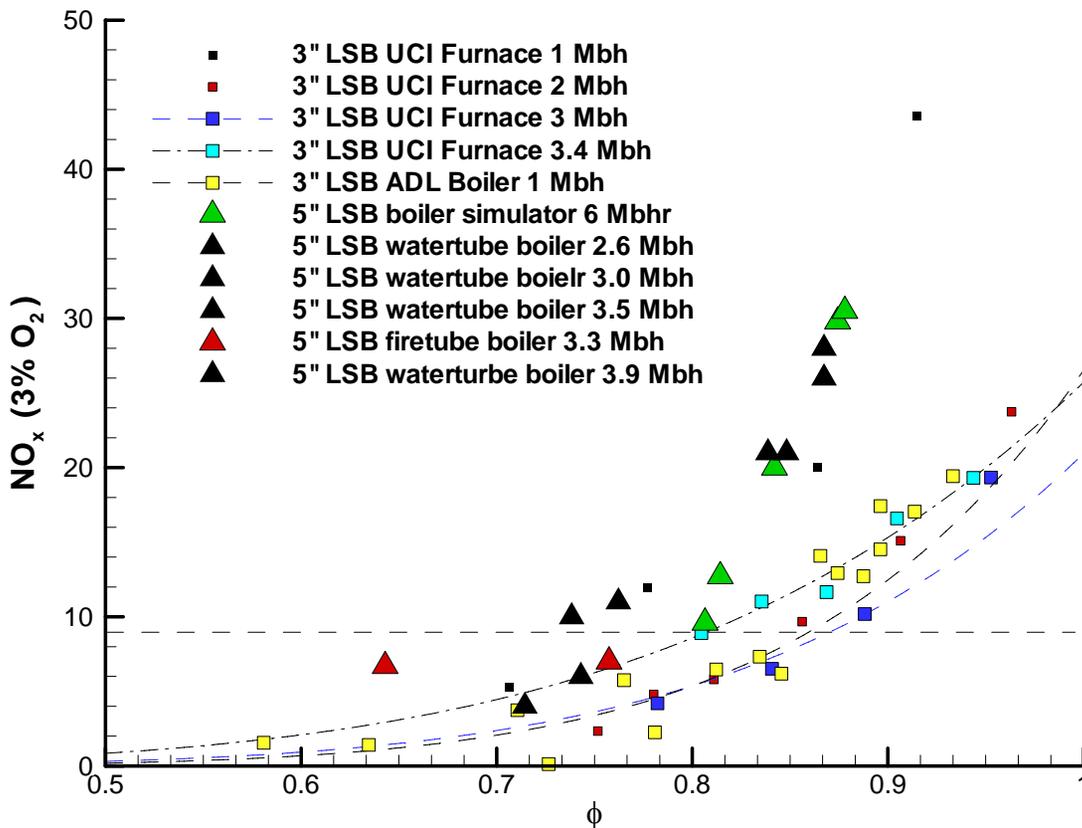


LSB Scaling Rules



- Keep swirl recess at 1 to 1.5 diameter
- Apply $0.4 < S < 0.55$ criterion
 - ▶ **Center-channel/burner ratio $0.5 < R < 0.6$**
 - Larger R increases drag thus blower power
 - ▶ **Vane angle between 37° to 45°**
 - Vane can be curved or straight, overlapping vanes increase turndown
 - ▶ **Use different screens to change S**
 - Screen geometry is not critical, other options available to change m
- Constant velocity scaling for power output
 - ▶ **Output power scaled by the square of the burner diameter**
- Optimum flame closure at 3 to 4 R_b

Applied $0.4 < S < 0.55$ Criterion to Scale LSB up to 16" and 50 MMBtu/hr



- **NO_x correlates primarily with air/fuel ratio**
 - ▶ Chamber geometry has some effect due to internal flow pattern and residence time
 - ▶ Cross < 9 ppm (3% O₂) threshold at 3% to 5.5% O₂
- **LSB flame is quiet and remains stable even at high excess air and FGR**

Basic Understanding Helped Practical Implementation

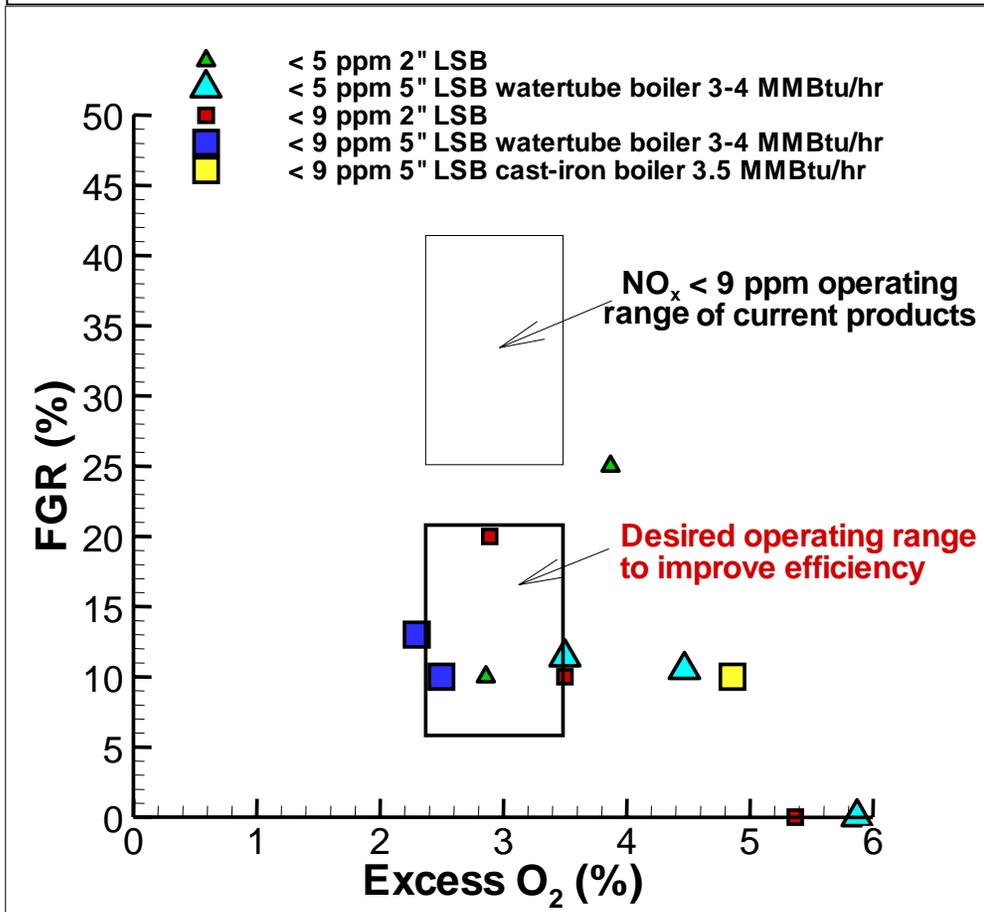
- **Applied theoretical knowledge on**
 - ▶ **Turbulence scaling, production, and dissipation**
 - ▶ **Flame temperature, flame speed and reaction chemistry**
 - ▶ **Combustion aerodynamics**
- **Identified, prioritized and resolved operational issues on**
 - ▶ **Placement of flame ignitor**
 - ▶ **Protocol to maintain flame stability during load change**
 - ▶ **Premixing requirement**
 - ▶ **Conditioning of flow supplied to the burner**

Maxon Identified Significant Economic and Technical Advantages of LSB

- **Design scales by governing equations**
 - ▶ A radical departure from experimentation approach
- **Size compatible to existing equipment**
- **Can be fabricated with no initial re-tooling or new patterns required - fewer parts from common materials**
- **Operates on conventional fuels (e.g. natural gas & propane) and uses flow controls designed for high NO_x burners**
- **Flame is not in contact with burner tip**
 - ▶ No thermal stresses to cause metal fatigue
- **Lower operational cost, and greater ease of operation, owing to a simple combustion process**

LSB Tested in Commercial Watertube & Firetube Boilers with External FGR

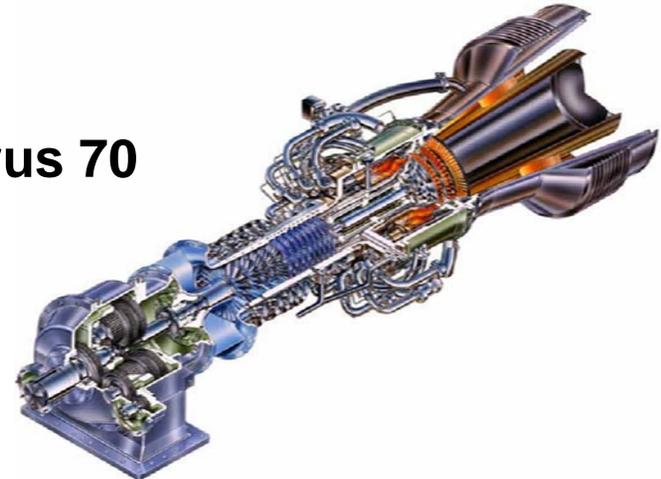
LSB points for < 9 & < 5 ppm NO_x (@3% O_2)



- Use blower and controls for the commercial boiler
- Demonstrated low NO_x at partial load
- In-chamber flow pattern alters NO_x formation
- LSB shows good promise for improving system efficiency

DOE-DER Low-emission Turbine Project with Solar Turbines

- **LSC to meet metrics for DOE-DER low emissions turbine program**
 - ▶ **< 5 ppm NO_x (@ 15% O₂)**
 - ▶ **Consideration for transition to back-up fuels**
 - ▶ **Durable for at least 8000 hours**
 - ▶ **No more than 10% cost add-on**
 - ▶ **No negative impacts on gas turbine performance**
- **Develop low-swirl injector for Solar Taurus 70 engine**
 - ▶ **≈7690 kW (10,310 hp) 16:1 compression ratio**
 - ▶ **3 – 4 % ΔP**
 - ▶ **Annular liner fitted with 12 injectors**

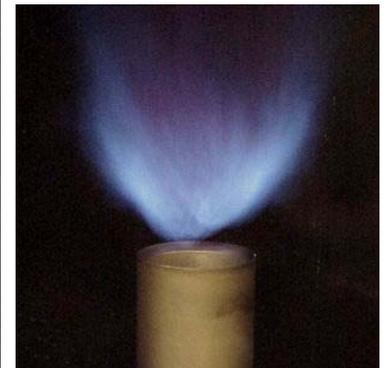
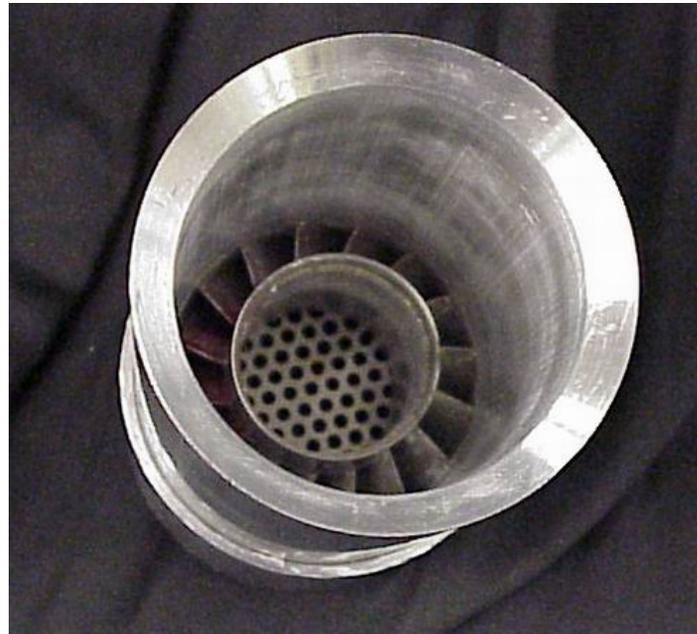
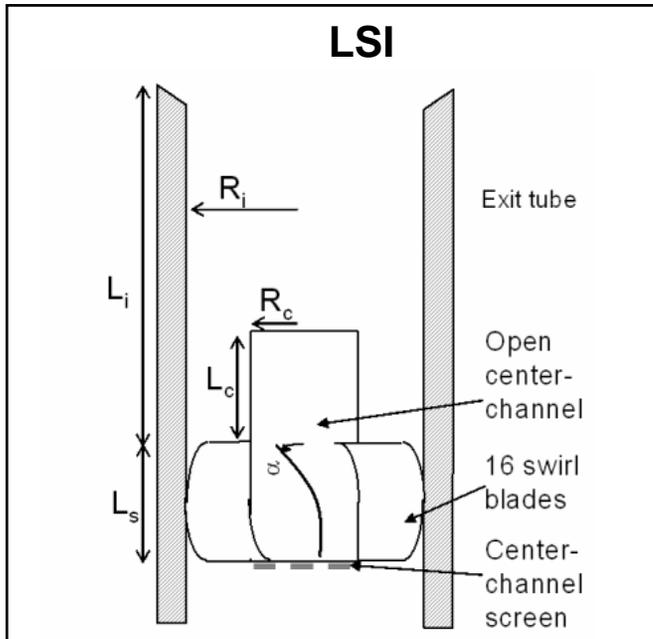


Developmental Steps of LSI for Solar Taurus 70

- Laboratory configuration and evaluation
- Single injector tests at high temperature and pressures
- Development of a fully functional LSI with pilot and premixer
- Atmospheric multi-injector tests in annular liner
- ~~Multi-injector pressurized tests in a developmental loop-engine~~
- Engine tests and optimization

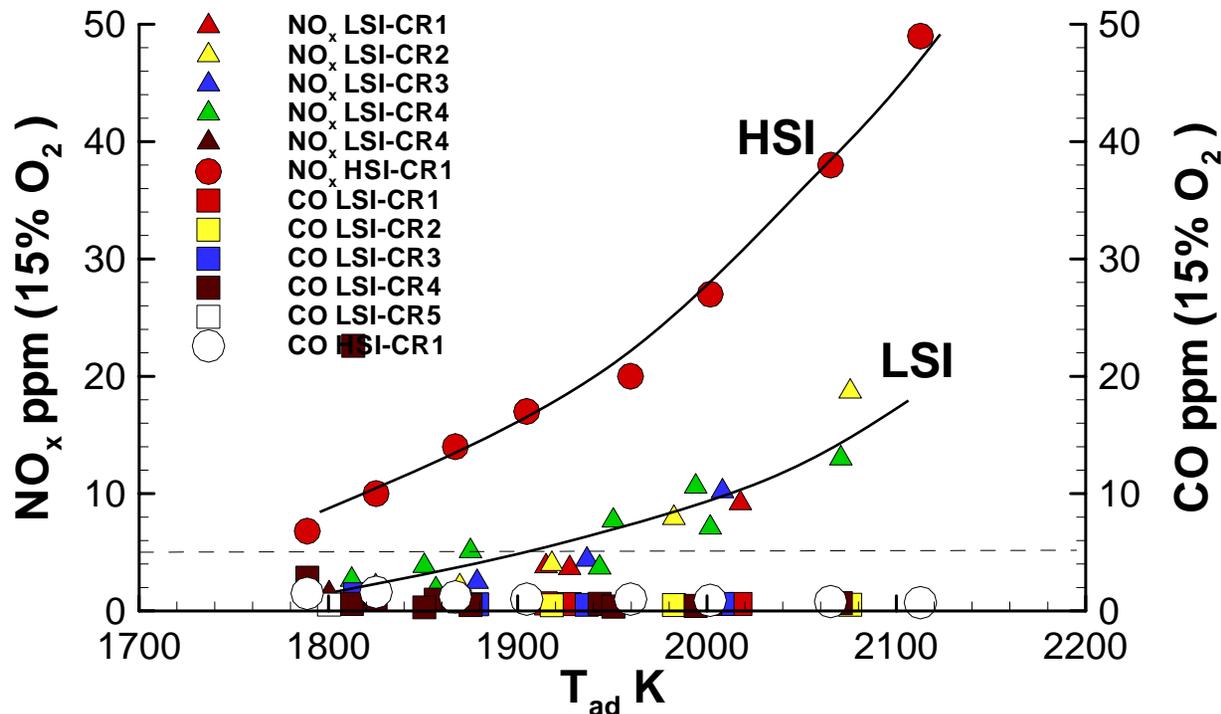
1st Low-swirl Injector (LSI) Prototype

- Utilize vane swirler for SoLoNOx Taurus 70
 - ▶ Configured in laboratory at STP and low U_0 (8 m/s)
 - Replace centerbody with perforated screen (38% open)
 - Followed guidelines from industrial burner development
 - $0.4 < S < 0.55$
 - 1.5 ratio between barrel length (swirler recess) and injector diameter



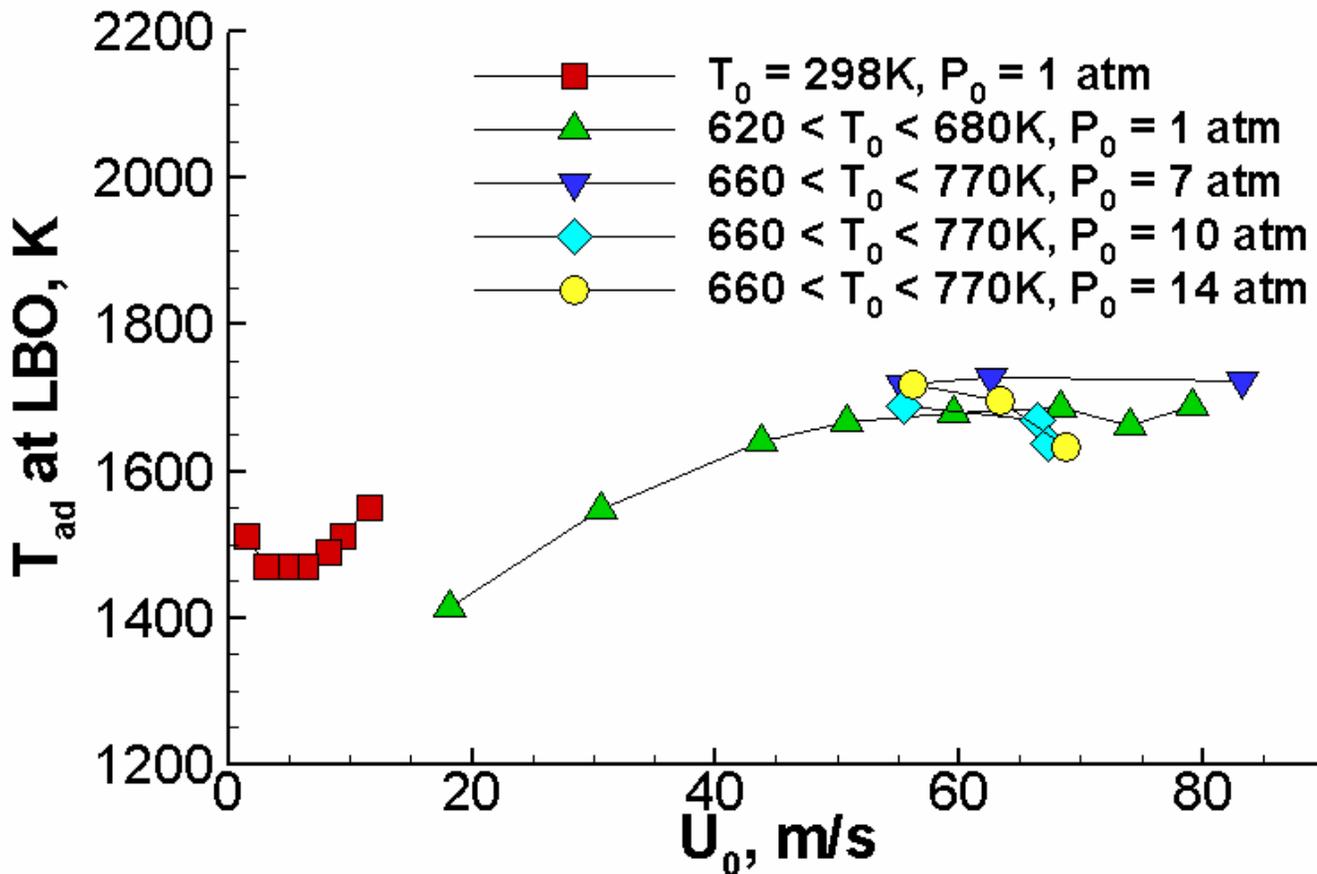
Single Injector Rig-Test Showed Ultra-Low Emissions

- Demonstrated low-swirl injector concept at full and partial loads ($500 < T_{in} < 900\text{F}$, $5 < P < 14 \text{ atm}$)
- NO_x emissions of LSI 60% lower than conventional DLN high-swirl injectors
- CO emissions well below acceptable limit
- Emissions same for +/- 10% and +/- 3% premixer



LBO of LSI unchanged at velocities up to 60% above design point

- LBO remains relatively insensitive to bulk flow velocity U_0



Development of an Engine-ready LSI Prototype

- **Confirm LSI operability within a typical engine cycle**
 - ▶ Design and testing of a fully functional LSI prototype with a premixer and a pilot
- **Configure the LSI to be “Drop-in” injector to replace SoLoNOx injector for Taurus 70**
 - ▶ Demonstrate engine readiness and low impact on engine performance
 - ▶ Evaluation criteria
 - Operability (light-off, loading & unloading protocol, response to off-design conditions)
 - Injector to injector interactions
 - Combustion oscillations
 - Emissions

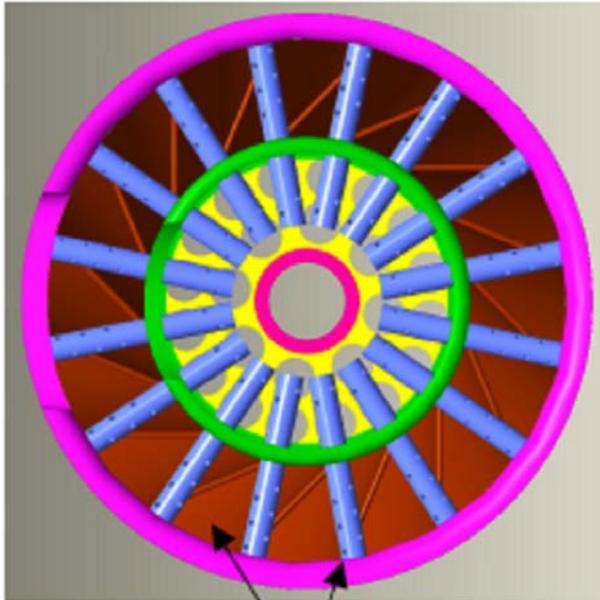
Integration of a Pilot for LSI

- **Embedded central pilot gives the best performance among several options**
 - Particle image velocimetry (PIV) measurements to optimize pilot tube size and assess effects on flowfield
 - Direct injection of pilot fuel to create a locally richer zone in the central non-swirling region
 - Flame burns in premixed mode from ignition to full load



LSI-2 with embedded pilot

Configured Simple Premixer



16 axial vanes & fuel spokes

- **Similar to SoLoNOx multi-spoke design**
 - ▶ Longer fuel tubes supplying both center channel and swirl annulus
- **Optimize to achieve desired homogeneity**
 - ▶ Varied the number and location of the injection holes
 - ▶ Radial uniformity +/- 20%, circumferential uniformity +/- 5%
- **Laboratory experiments (at STP) to verify functionality**
 - ▶ Comparison of flowfields and flame positions with well mixed cases

LSI-3 for Taurus T70

Solar Taurus 70 SoLoNOx
Injector/premixer assembly



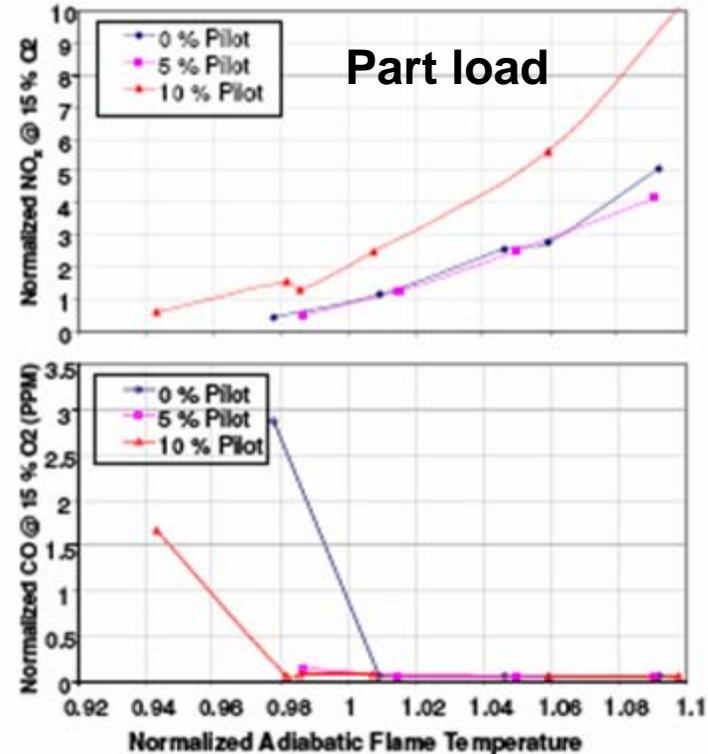
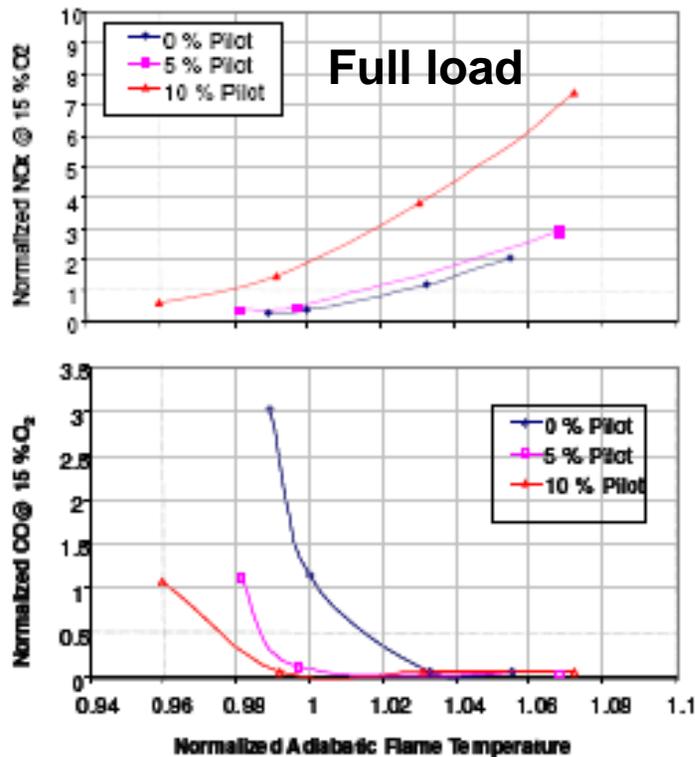
Solar Taurus 70 LSI
injector/premixer assembly



- **LSI-3 built from SoLoNOx swirler**
 - Significant savings in engineering and fabrication
- **Same overall size and mounting configuration as T70 SoLoNOx injector**
- **Built 15 injectors**
 - Selected two at random for baseline performance tests
 - 12 used for engine test

Baseline Performance of LSI-3 from Single Injector Tests

- 5% pilot offers ultra-low NO_x and extends LBO
- $\text{NO}_x < 2.5$ ppm
- CO well below acceptable levels
- 30% pilot further extends LBO

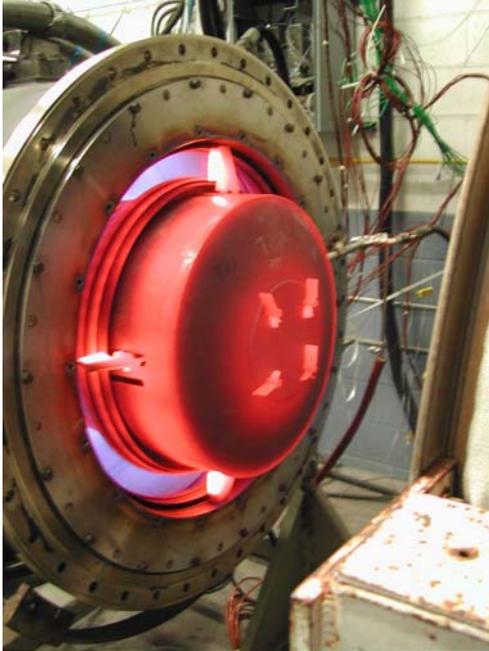


LSI Does Not Experience Significant Material Heating Due to Lifted Flame



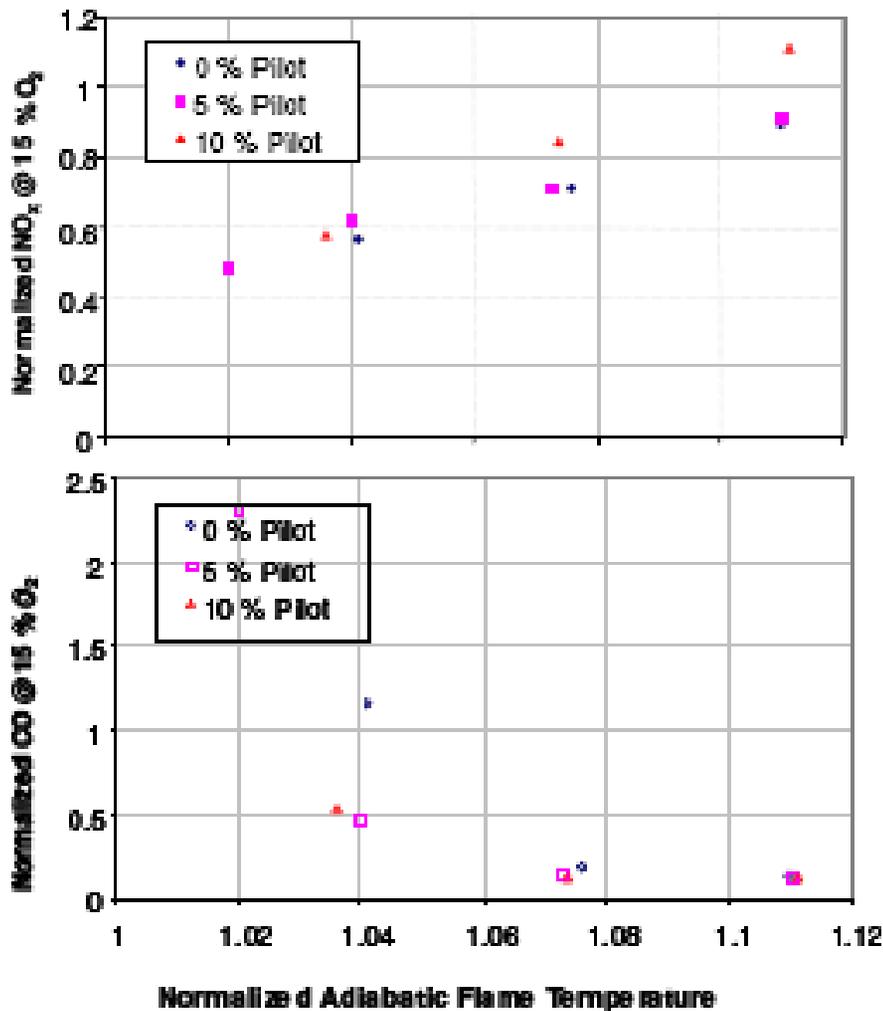
- Temperature sensitive paint tests of LSI-3 show most regions experience temperatures below 915F for $T_0 = 800\text{F}$
- Represents a major benefit of the technology
 - ▶ Maintaining low material temperature without requiring active cooling
 - ▶ prolong service life

Multi-Injectors Atmospheric Annular Liner Tests



- 12 LSI-3 fitted to a T70 annular liner and evaluated at simulated partial and full load with 0, 5 and 10% pilot.
- Circumferential and radial temperature distributions were within the acceptable limits
- LSI-3 showed excellent light-around characteristics with no indication of combustion harmonics or injector to injector interactions

LSI-3 Emissions in Atmospheric Annular Liner



- Trends similar to single injector tests
- Skipped testing in developmental loop engine and proceed directly to engine tests
- **1st round of LSI-3 tests in T70 engine complete June 2006**
 - ▶ Engine NO_x emissions identical to rig test results

LSI Meeting Metrics of DOE-DER Low Emissions Turbine Program

- **Robust technology offering significant savings in development, fabrication, operating and maintenance costs**
 - ▶ Simple scalable design built from existing parts
 - ▶ Configure by laboratory experiments without invoking extensive CFD design iterations
 - ▶ No special requirements for materials and control
 - ▶ Lifted flame induces minimal thermal stresses & metal fatigue
- **Exceptional performance**
 - ▶ $\text{NO}_x < 5 \text{ ppm}$ at T_{PZ} away from blowoff
 - ▶ No flame shift or flashback with load change
 - ▶ Flame robust to withstand large swing in inlet conditions
 - ▶ Vulnerability to oscillations no different than SoLoNOx
 - ▶ Emissions not sensitive to small degree of mixture inhomogeneity
 - ▶ **Lower ΔP for potential efficiency improvement**

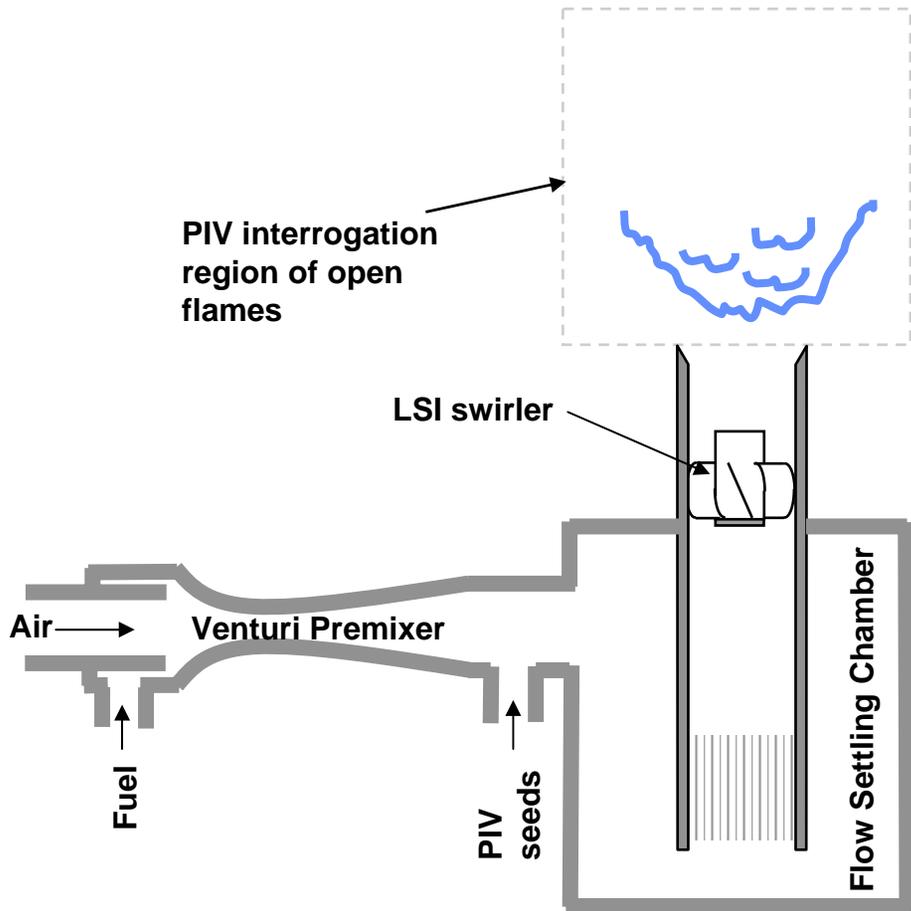
Part 3

LSC adaptation to large scale turbines for natural gas, syngas, and hydrogen fuels

Obtaining an Analytical Model for Low-swirl Combustion

- **Apply Particle Imaging Velocimetry (PIV) to measure instantaneous velocity vectors within an area of 13 x 13 cm**
 - ▶ Facilitates the collection of a large amount of flowfield data
- **Characterize flowfield and flame behavior as function of:**
 - ▶ swirl number
 - ▶ fuel type
 - ▶ fuel air ratio
 - ▶ bulk flow velocity
- **Define key parameters that characterize the flowfield**
- **Develop analytical model for the relationships between the flame and flowfield**
 - ▶ Basis for scaling and adaptation guidelines
 - ▶ Less reliance on computational fluid mechanics (CFD)
 - ▶ **Reduce uncertainties and increase predictability**

Apparatus, Diagnostics & Analysis

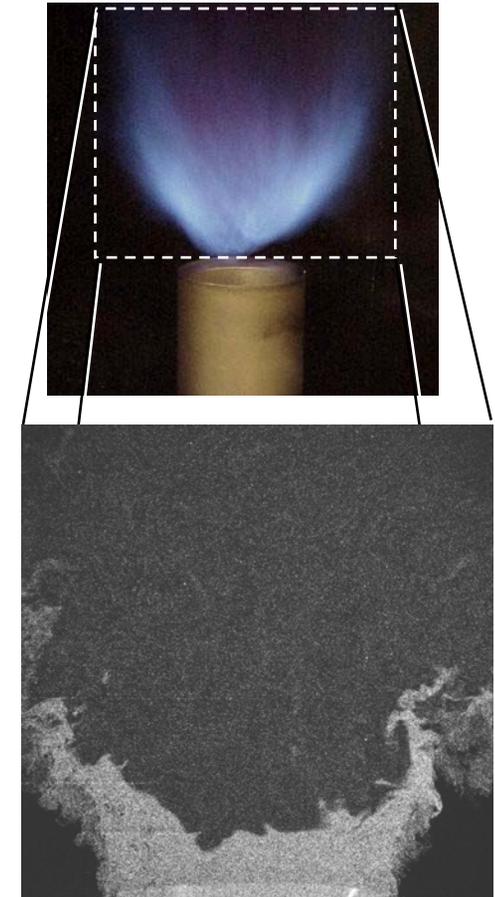


- LSI mounted to the plenum and premixer of an industrial burner
- Applied PIV to atmospheric open flames
 - ▶ Previous development with Solar demonstrated relevancy of open flame experiments
- Deduced mean, rms velocities, Reynolds stresses & turbulent flame speeds

Experimental Conditions

- Lean flames burning single and dual-component fuels with a range of Wobbe indices
- Operating regimes for each fuel defined by LBO and emissions
- Varied bulk flow velocity U_0 from 7 to 22 m/s

Fuel Composition	T_{ad} at $\phi = 1$ K	S_L at $\phi = 1$ m/s	S_L at $T_{ad} = 1800K$
CH_4	2230	0.39	0.17
C_2H_4	2373	0.74	0.23
C_3H_8	2253	0.45	0.22
0.5 CH_4 / 0.5 CO_2	2013	0.20	0.12
0.6 CH_4 / 0.4 N_2	2133	0.31	0.16
0.6 CH_4 / 0.4 H_2	2258	0.57	0.22
H_2	2535	2.4	0.61
0.5 H_2 / 0.5 N_2	2056	1.2	0.61

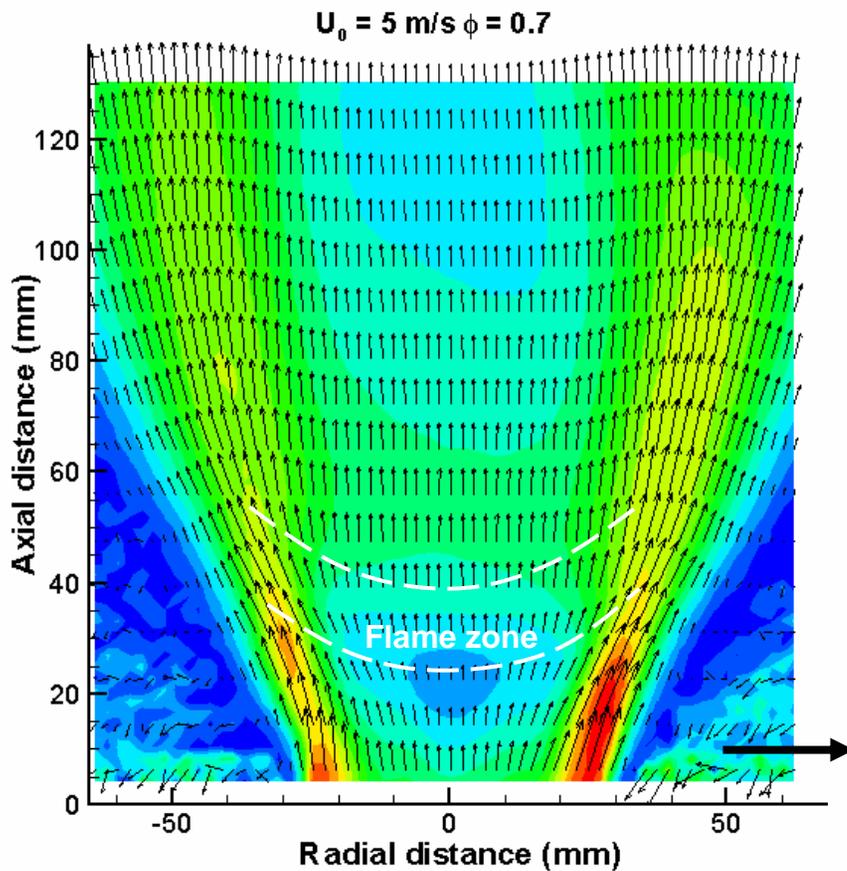


Raw PIV image showing wrinkled premixed turbulent flame structures

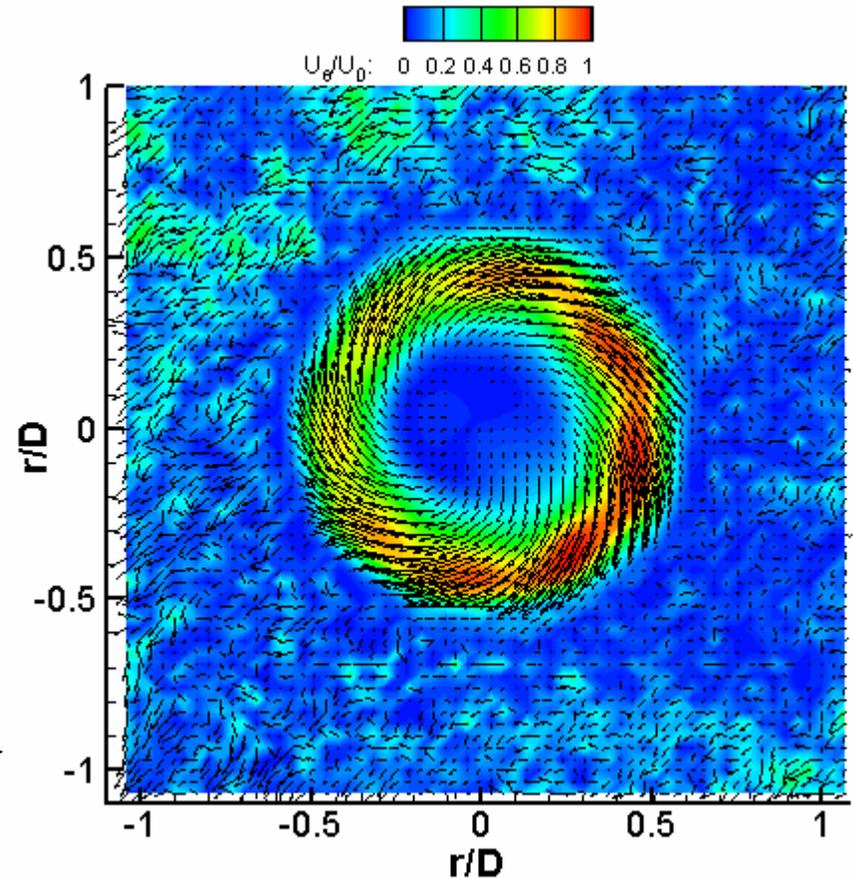
Hydrocarbon Flame Results

Flowfield of LSC Shown by Particle Image Velocimetry

- Mean velocity vectors on axial plane

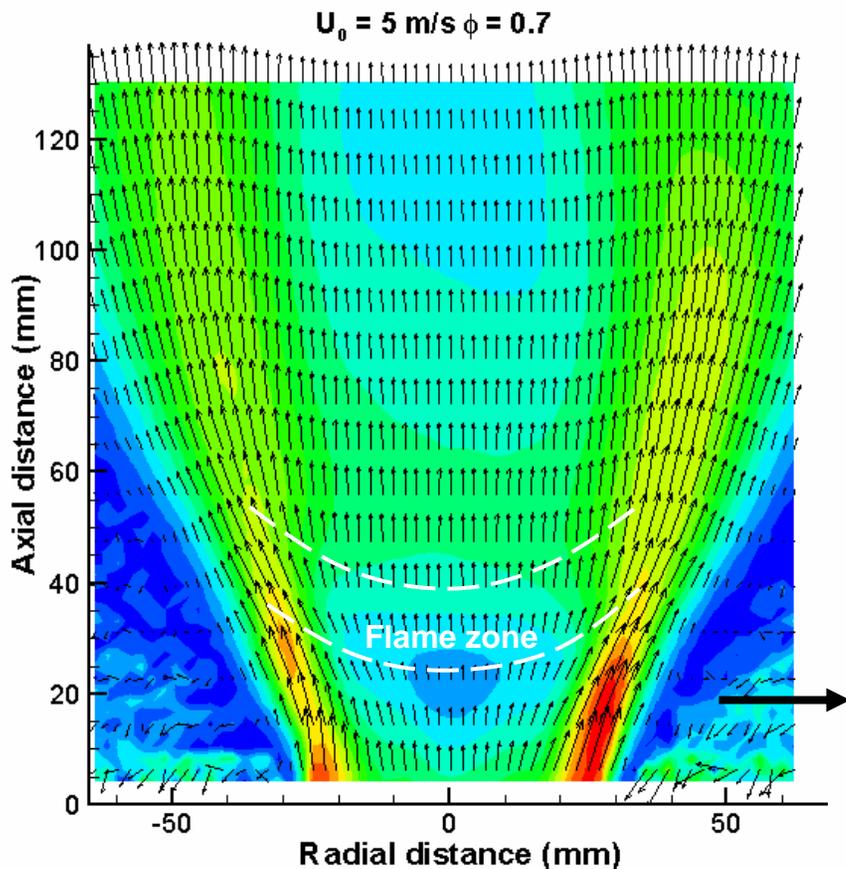


- Mean velocity vectors on cross-plane

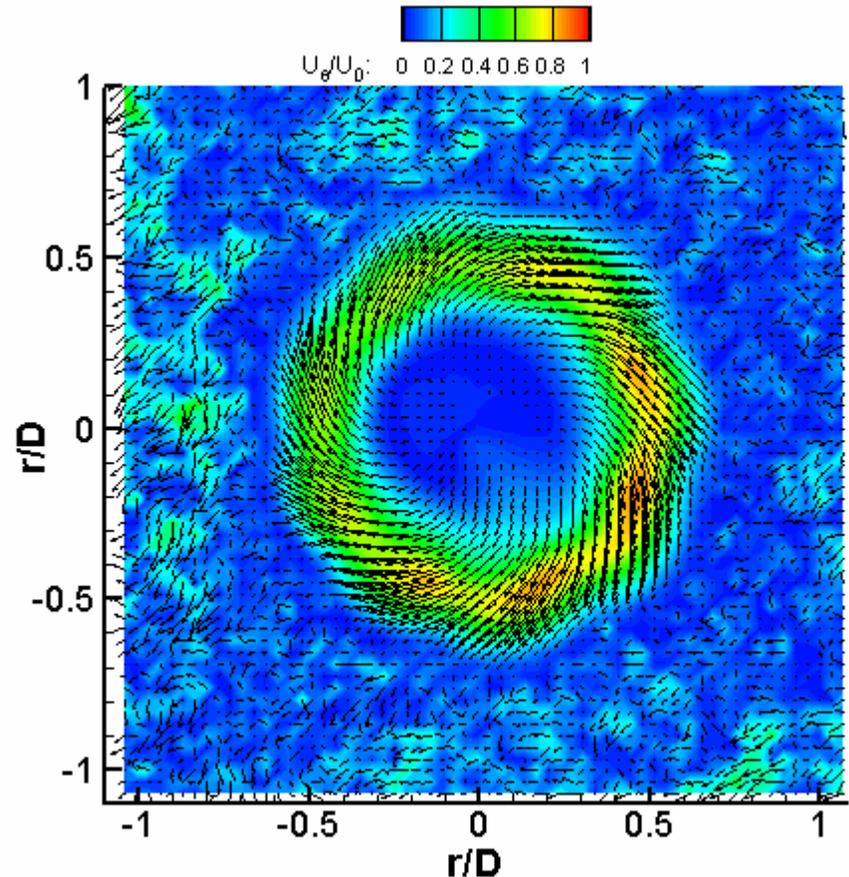


Flowfield of LSC Shown by Particle Image Velocimetry

- Mean velocity vectors on axial plane

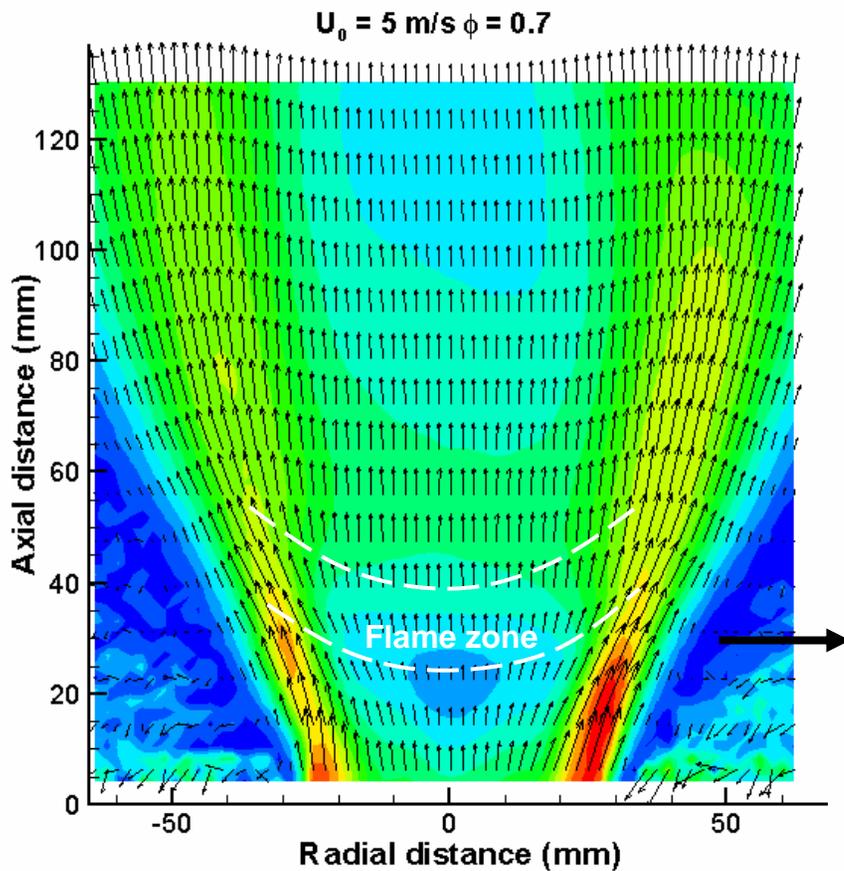


- Mean velocity vectors on cross-plane

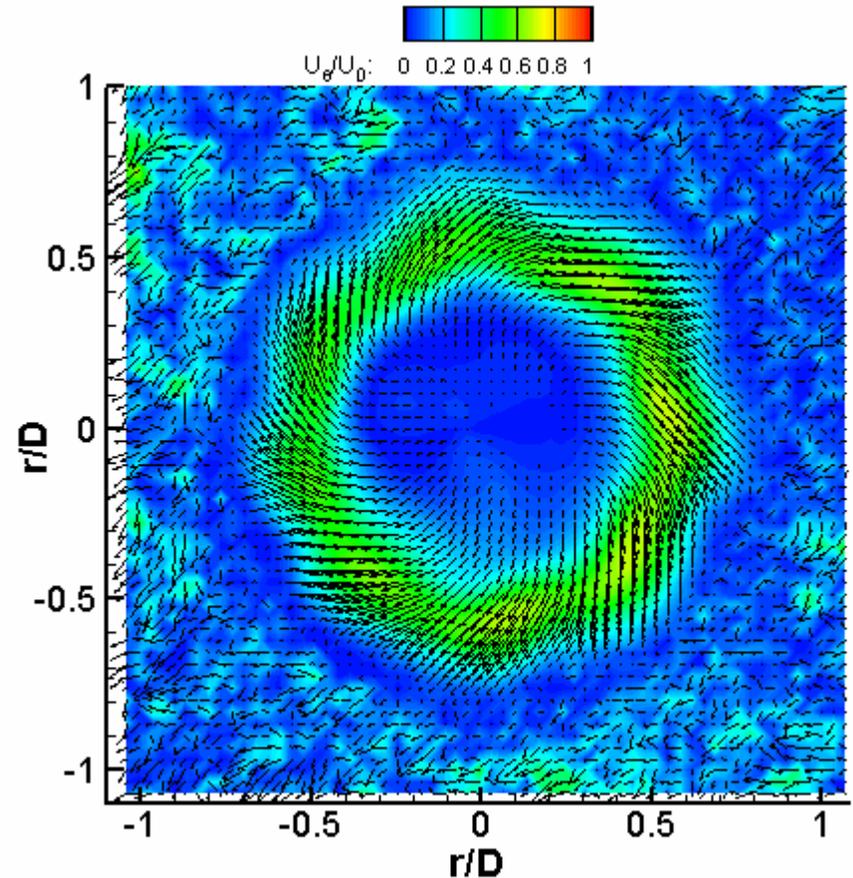


Flowfield of LSC Shown by Particle Image Velocimetry

- Mean velocity vectors on axial plane

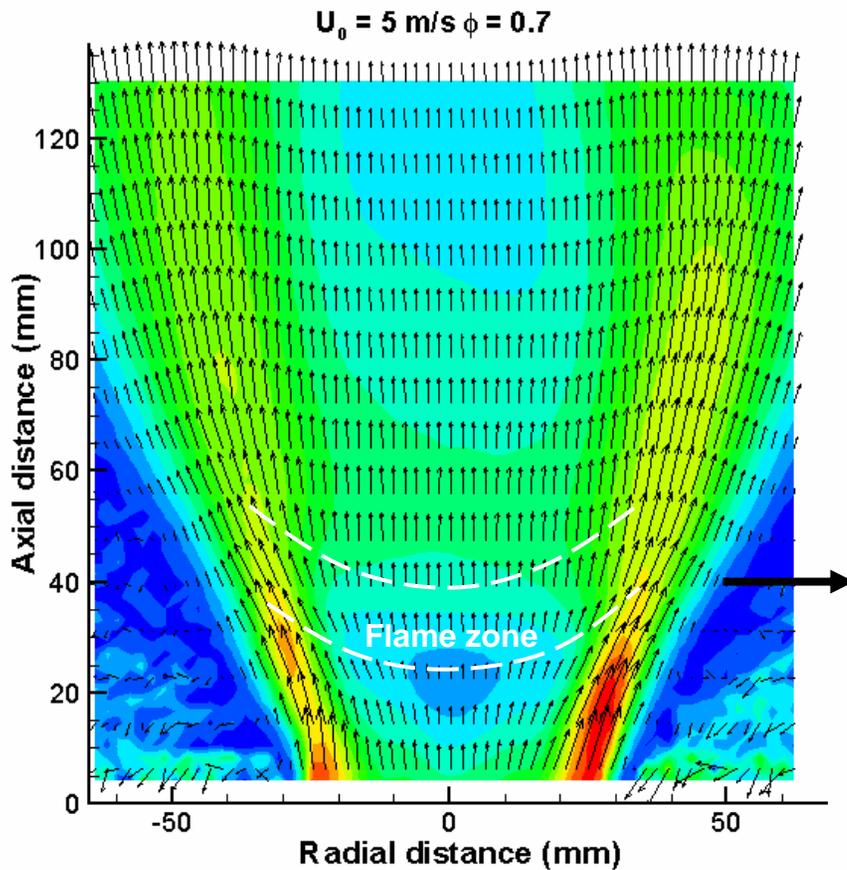


- Mean velocity vectors on cross-plane

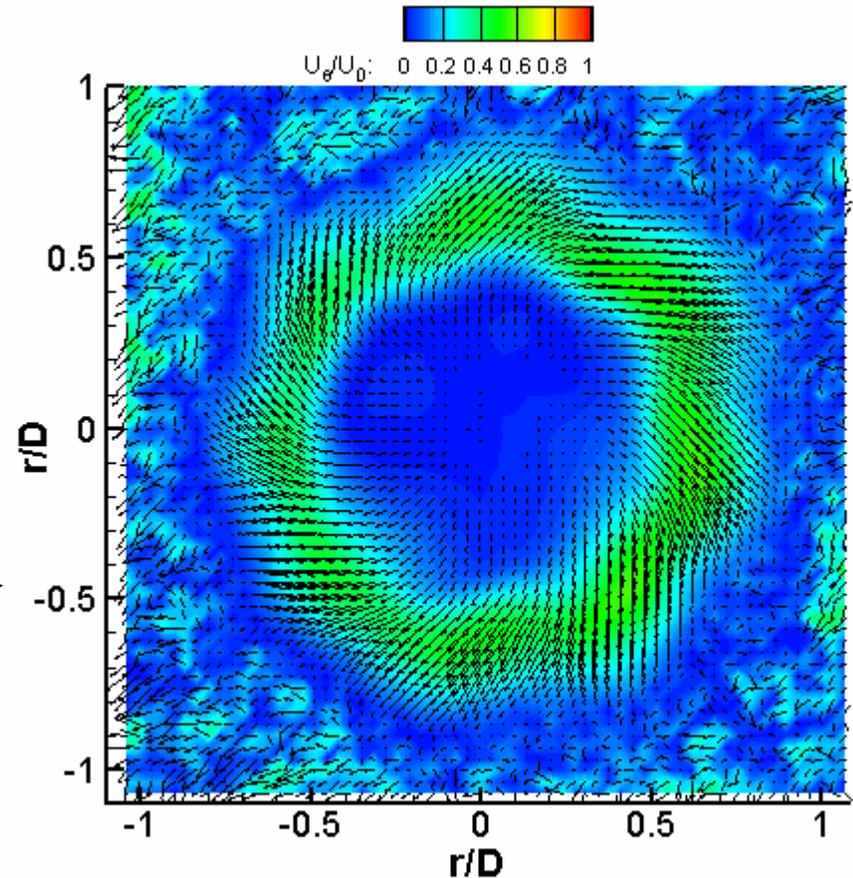


Flowfield of LSC Shown by Particle Image Velocimetry

- Mean velocity vectors on axial plane

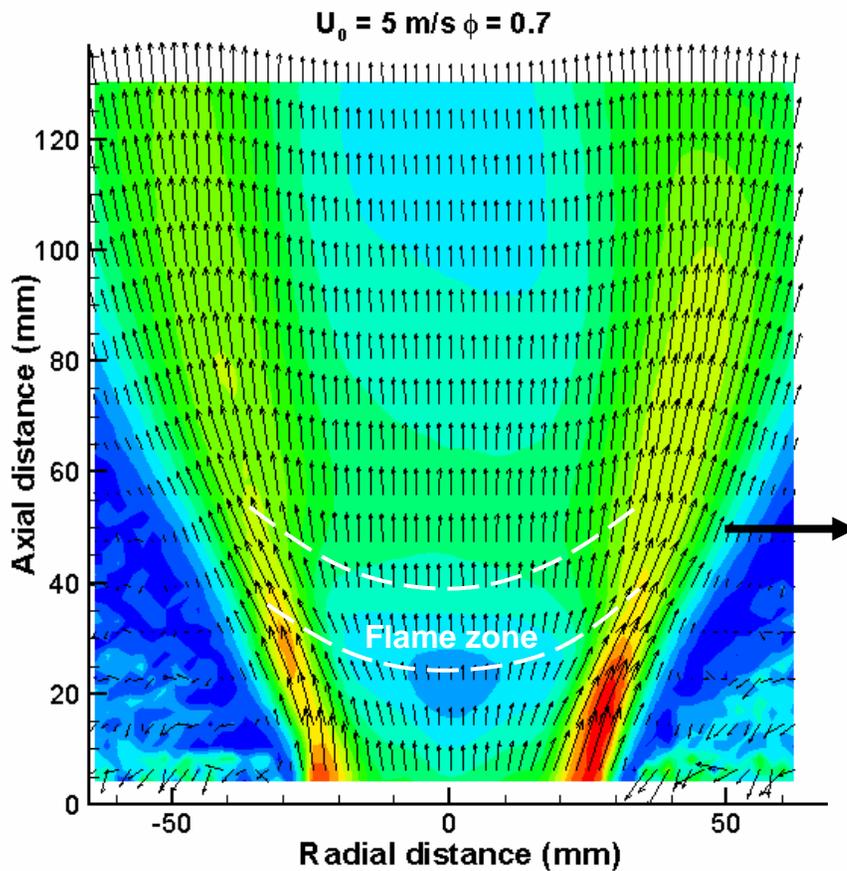


- Mean velocity vectors on cross-plane

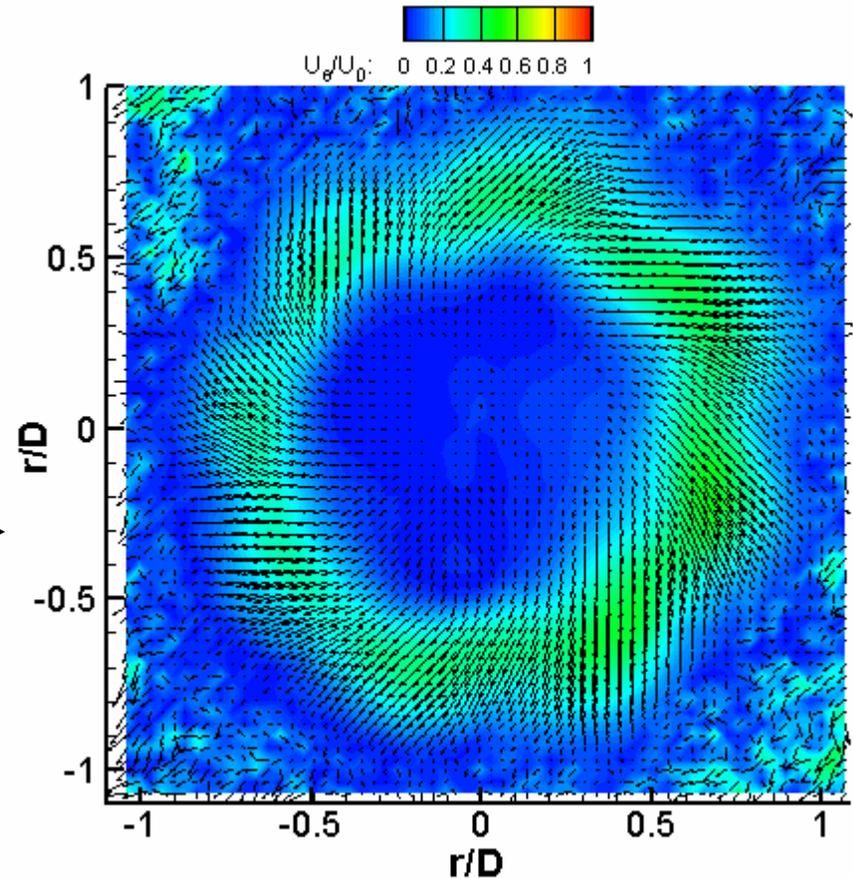


Flowfield of LSC Shown by Particle Image Velocimetry

- Mean velocity vectors on axial plane

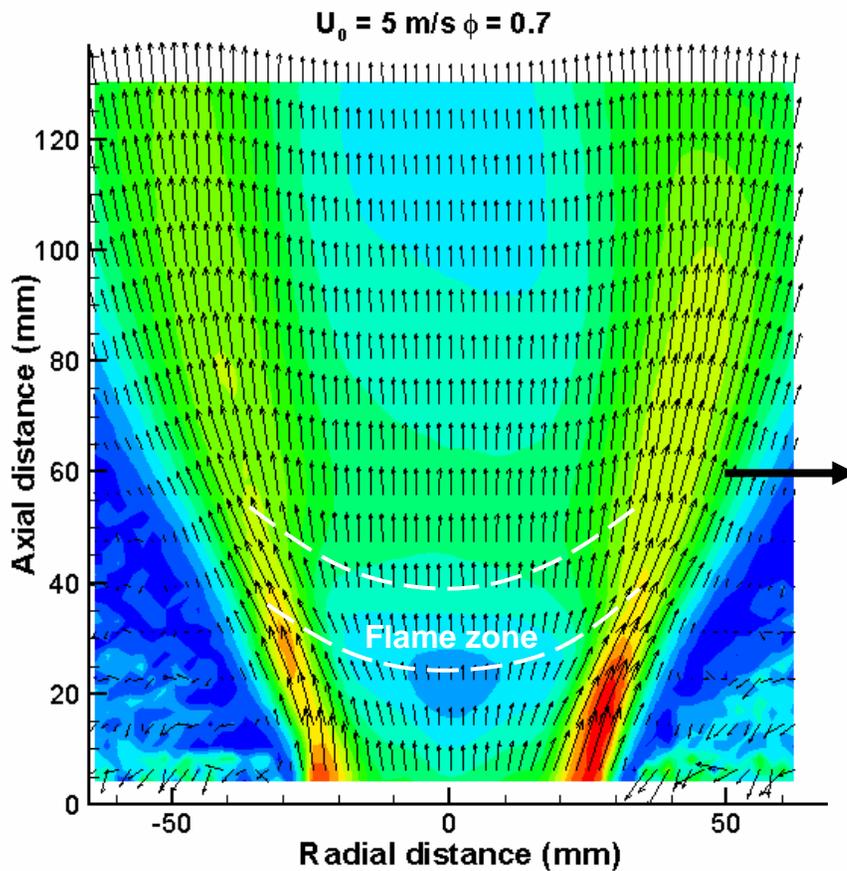


- Mean velocity vectors on cross-plane

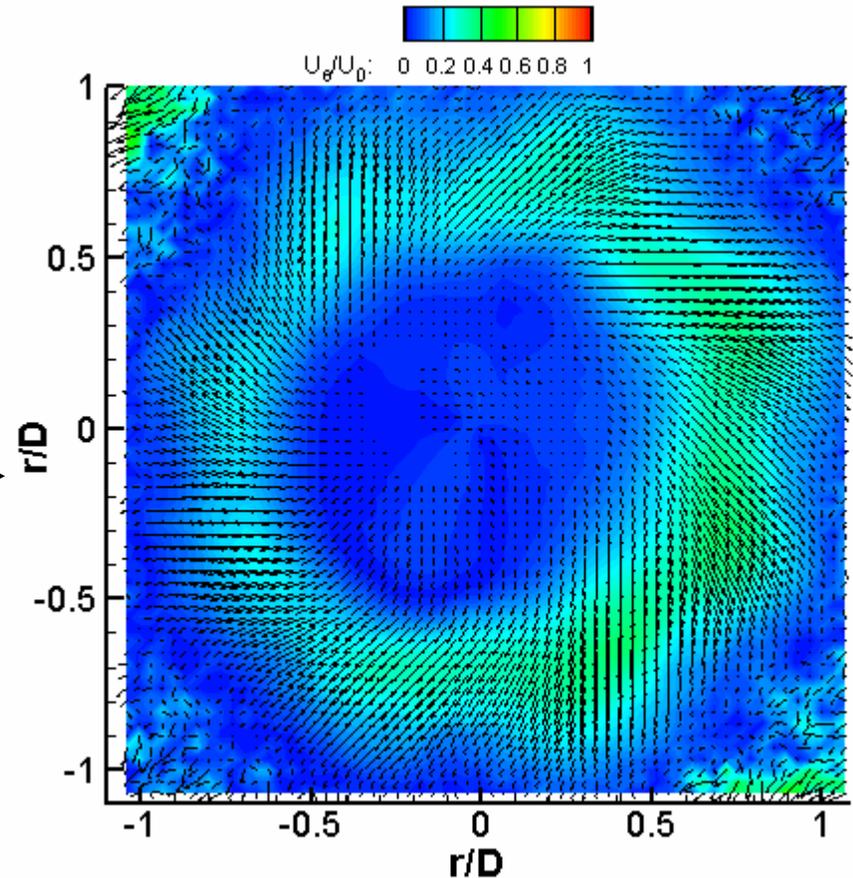


Flowfield of LSC Shown by Particle Image Velocimetry

- Mean velocity vectors on axial plane

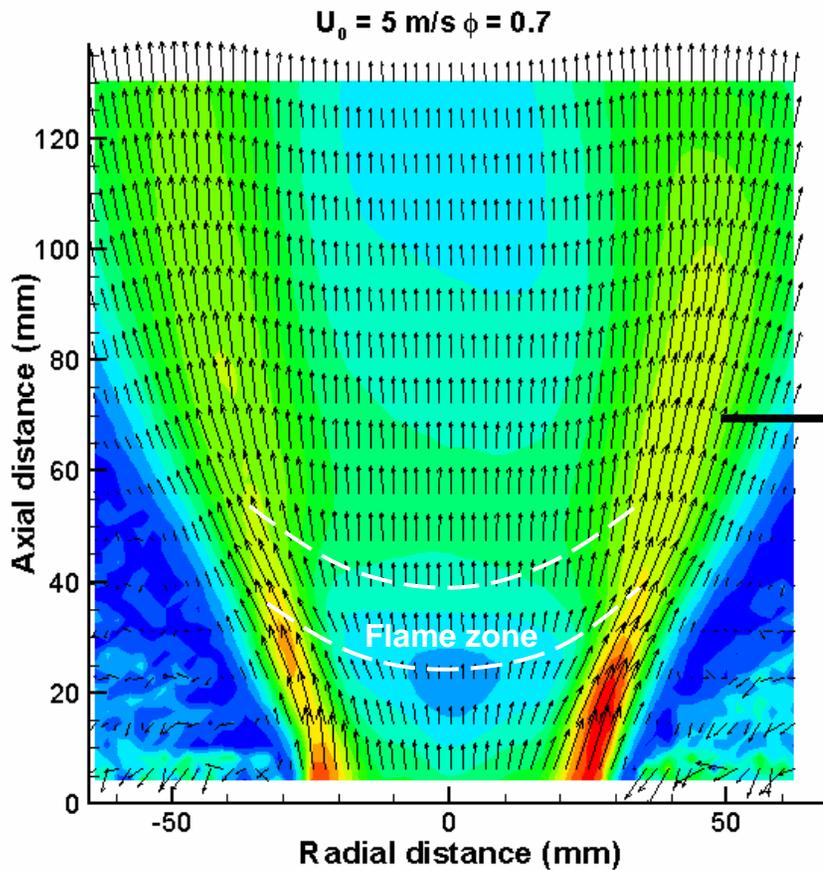


- Mean velocity vectors on cross-plane

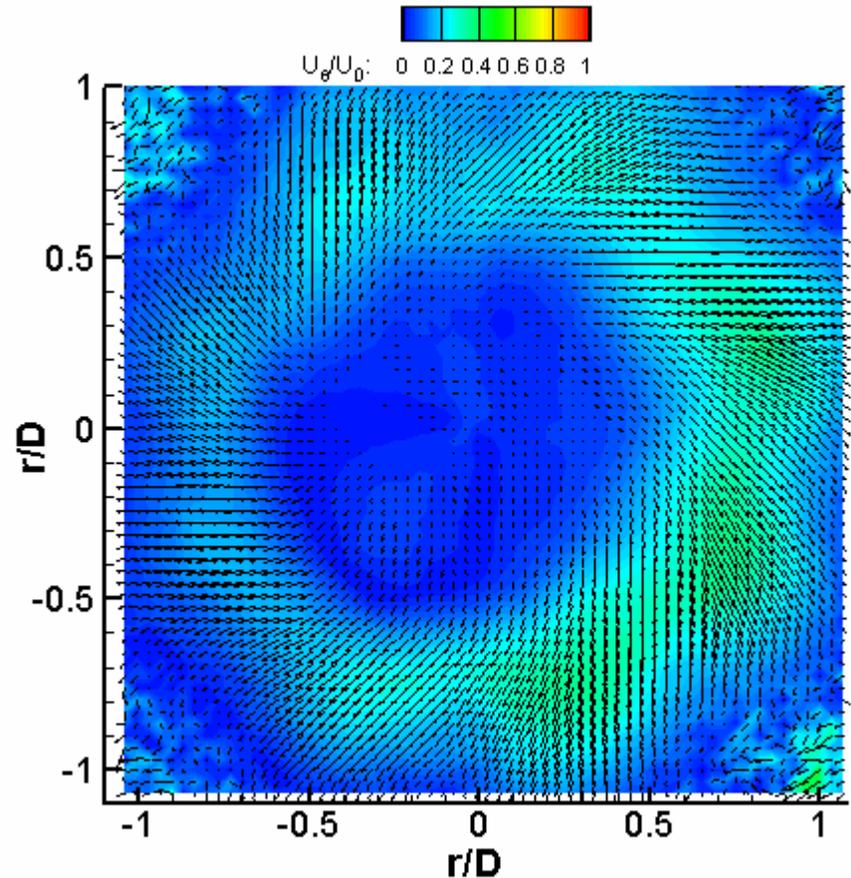


Flowfield of LSC Shown by Particle Image Velocimetry

- Mean velocity vectors on axial plane

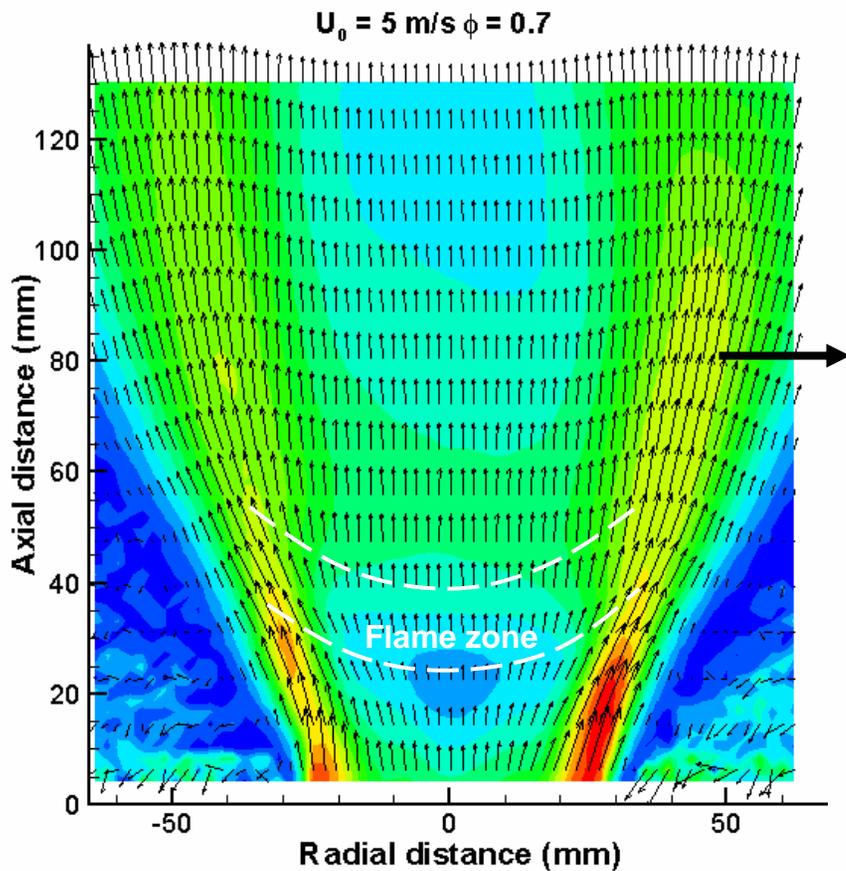


- Mean velocity vectors on cross-plane

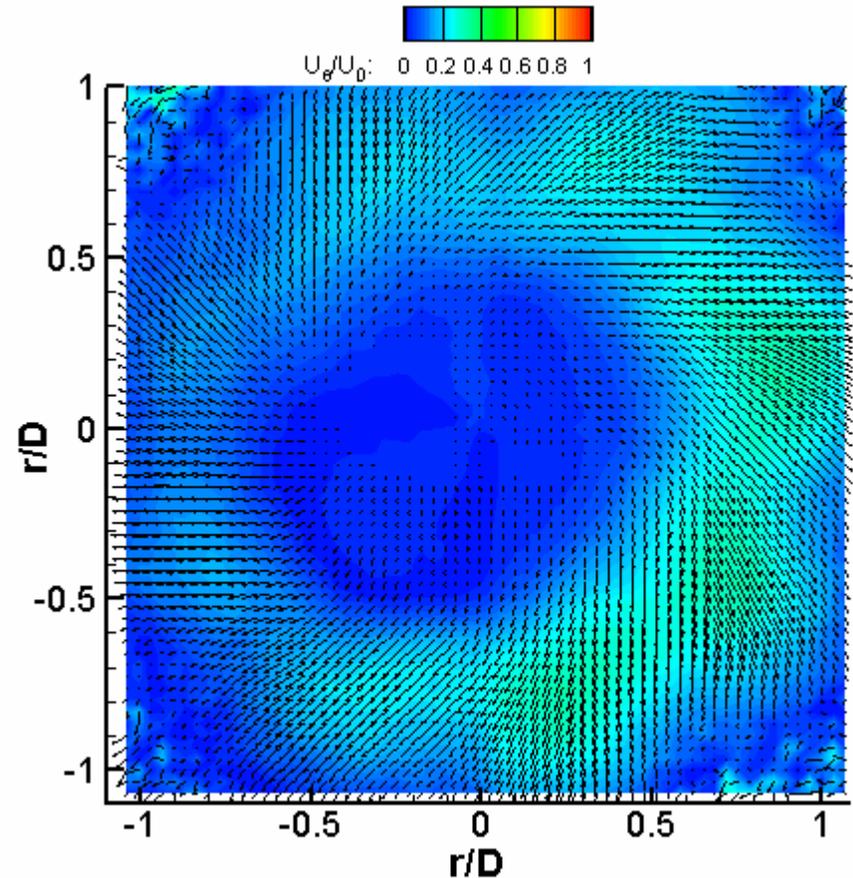


Flowfield of LSC Shown by Particle Image Velocimetry

- Mean velocity vectors on axial plane

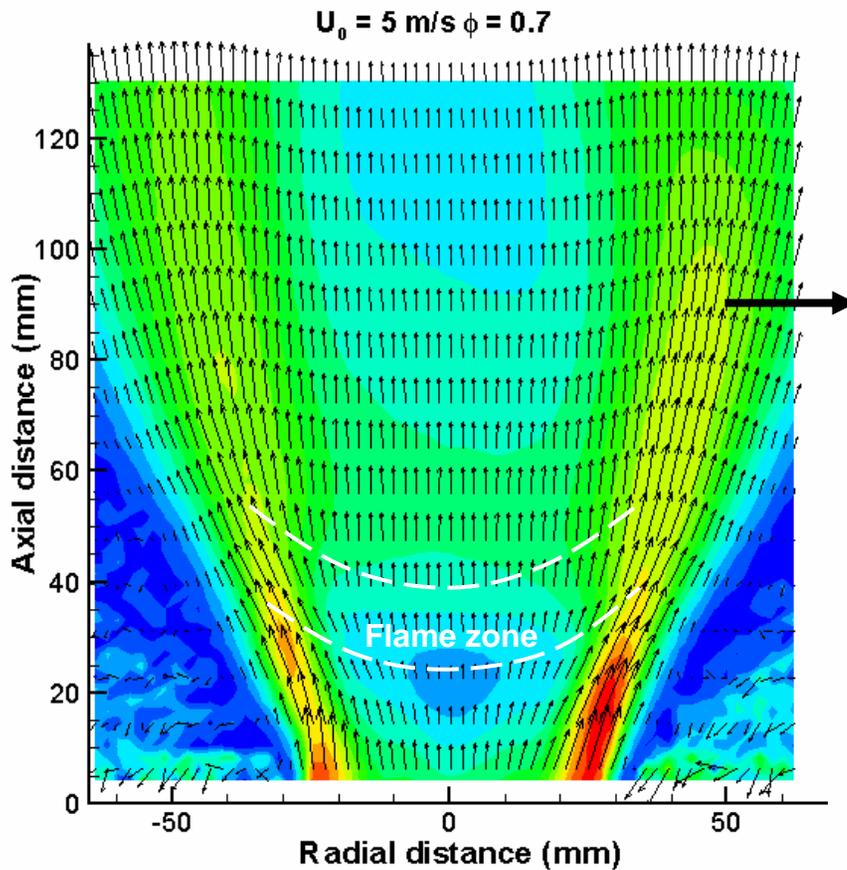


- Mean velocity vectors on cross-plane

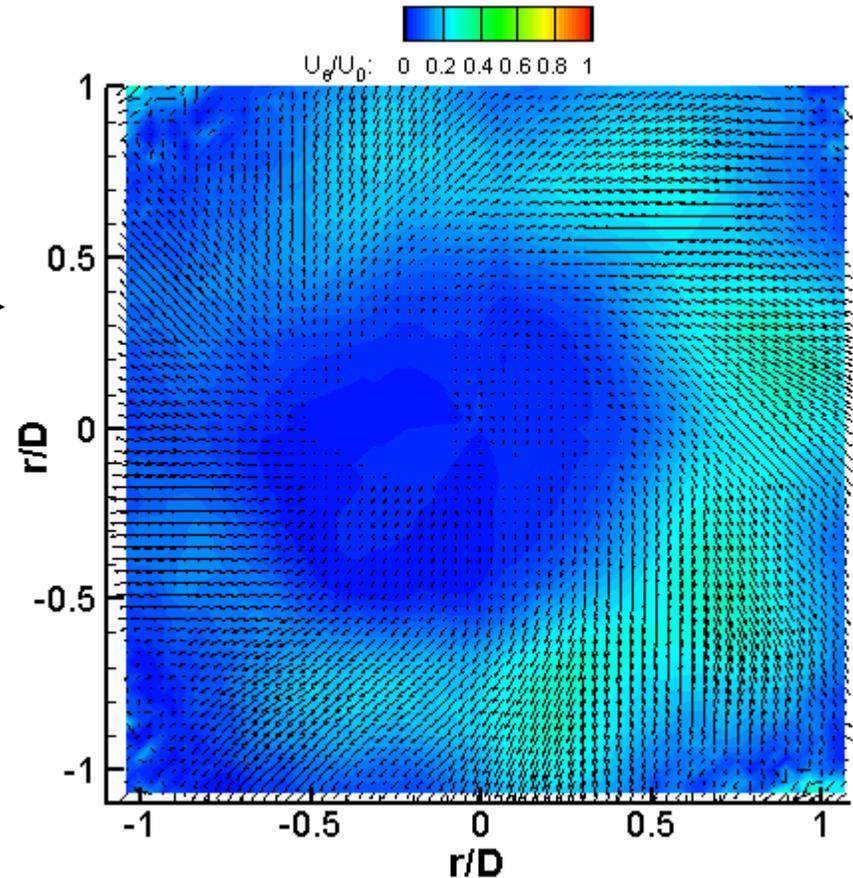


Flowfield of LSC Shown by Particle Image Velocimetry

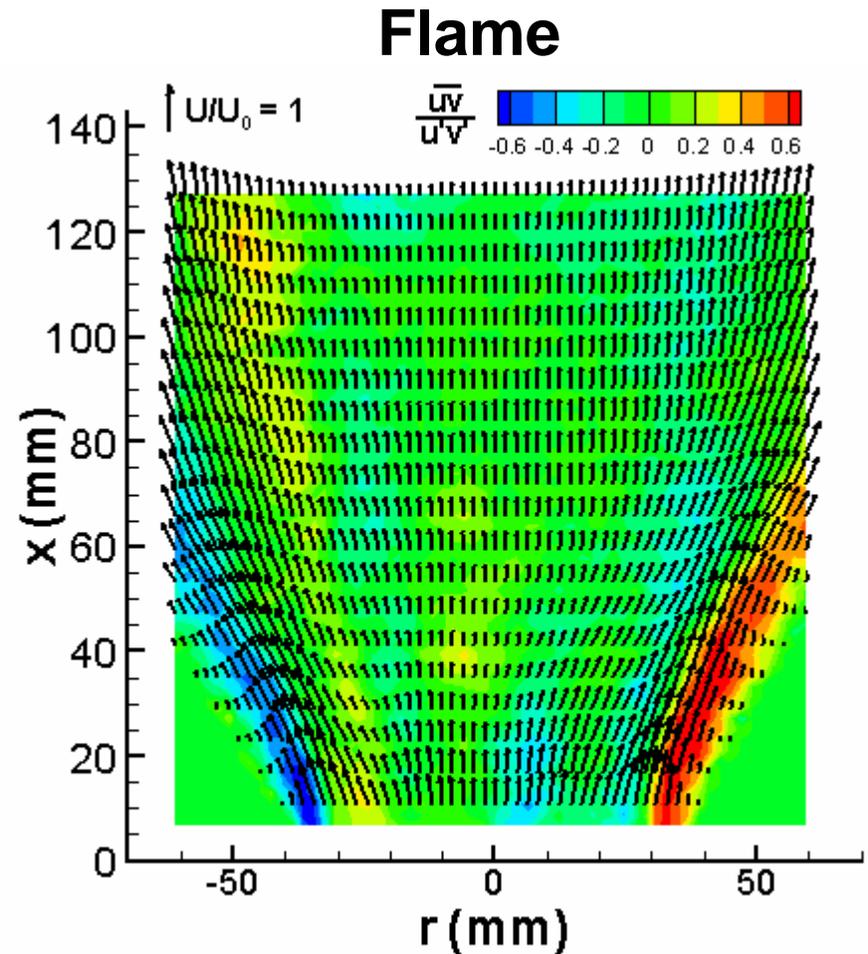
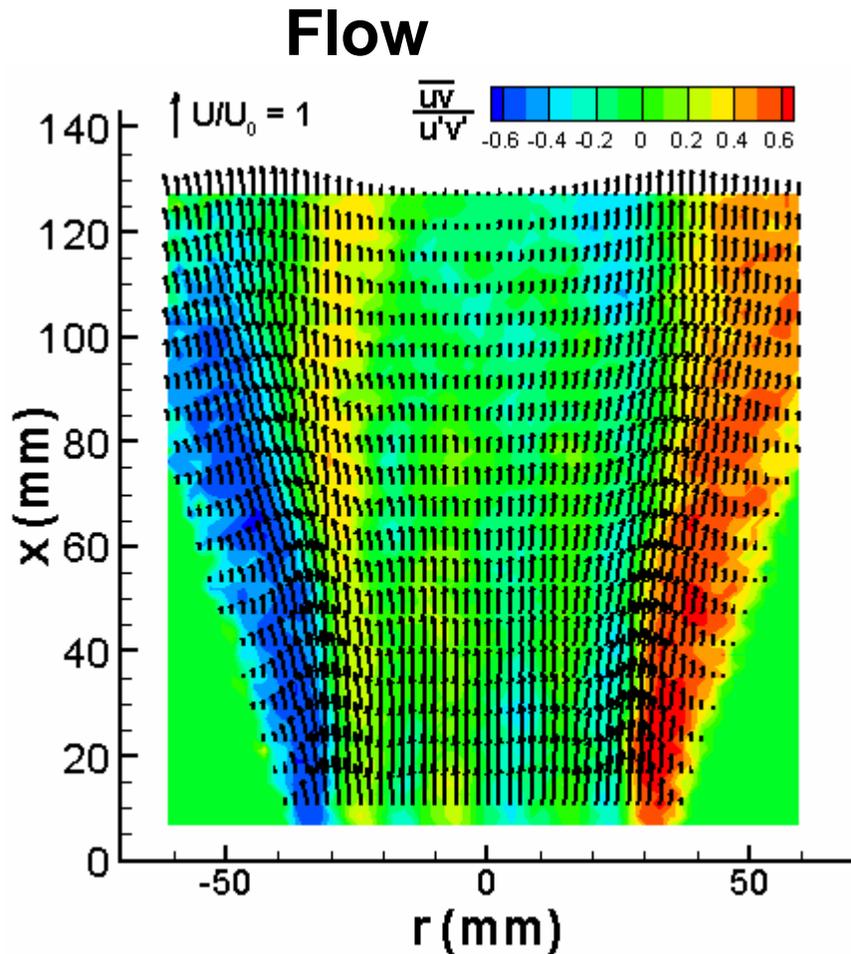
- Mean velocity vectors on axial plane



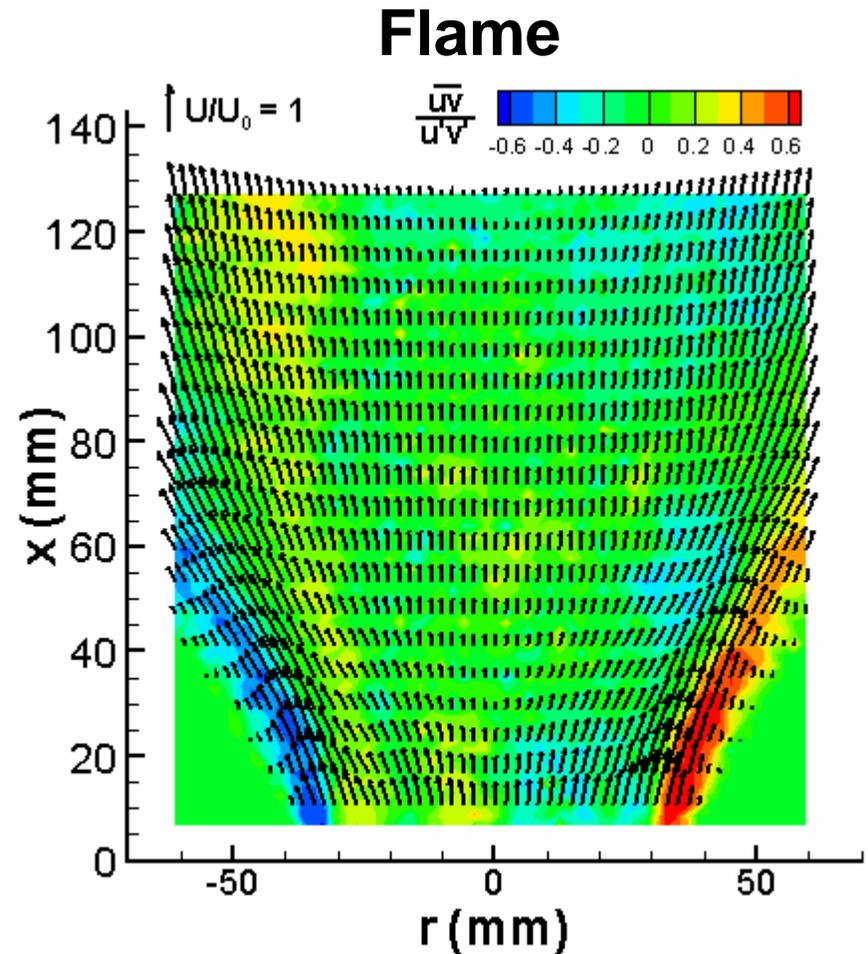
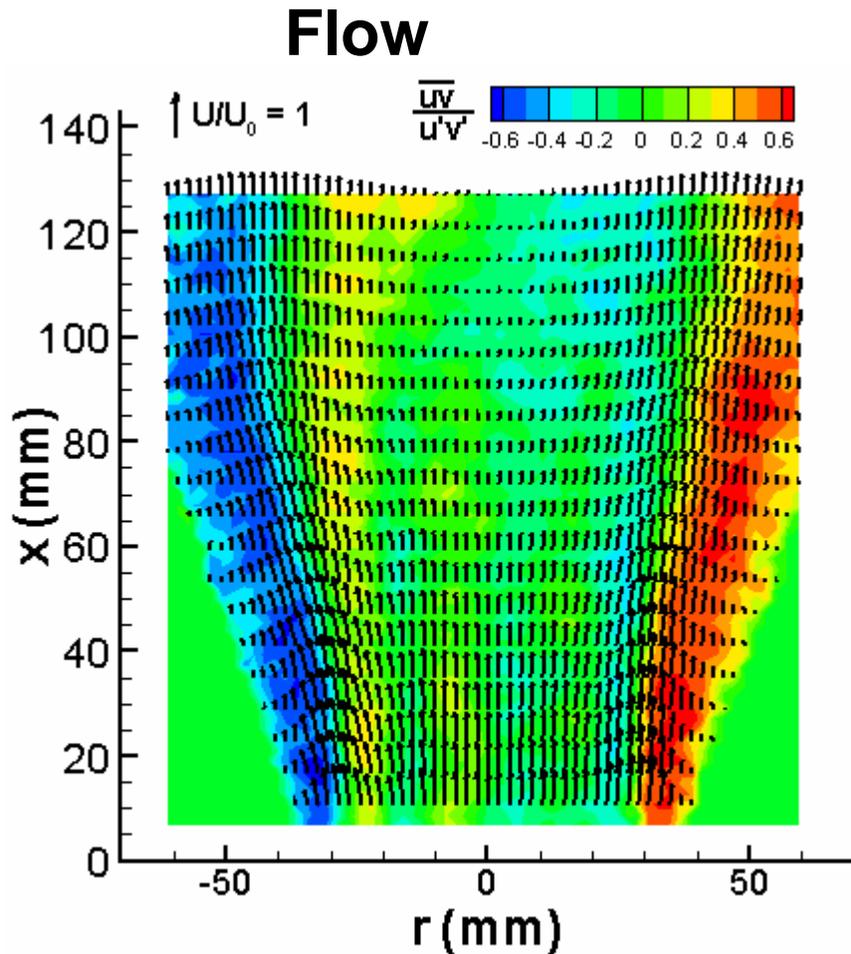
- Mean velocity vectors on cross-plane



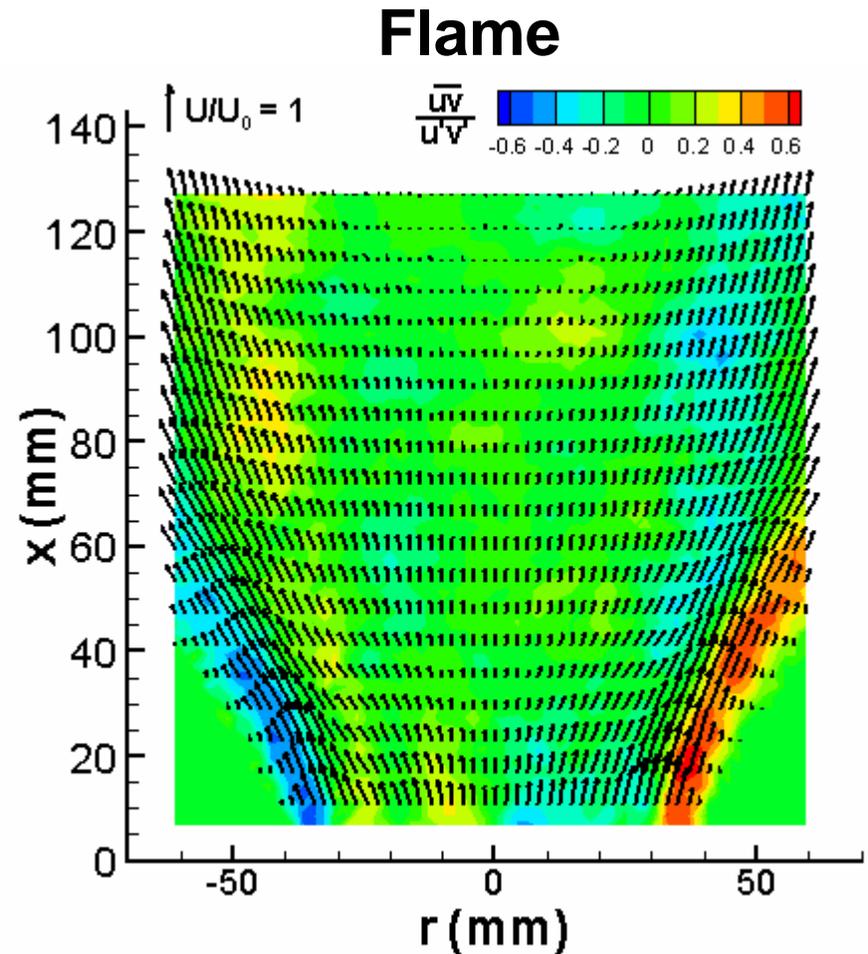
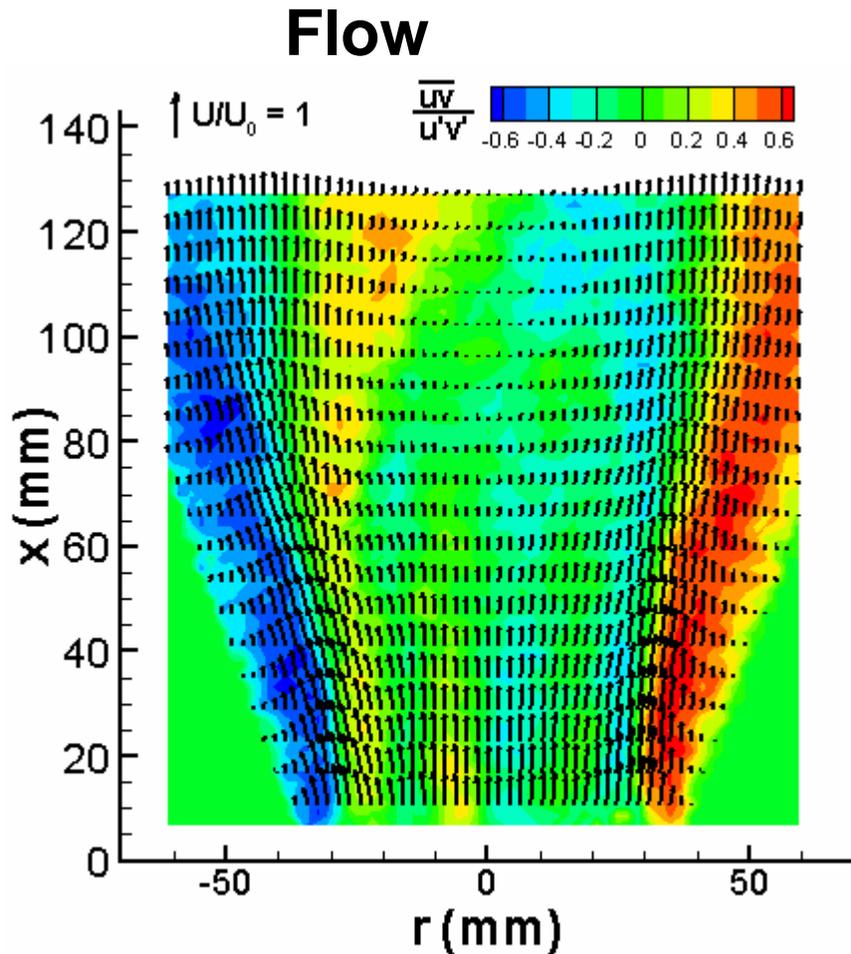
Normalized mean vectors and Reynolds stress at $U_0 = 7$ m/s



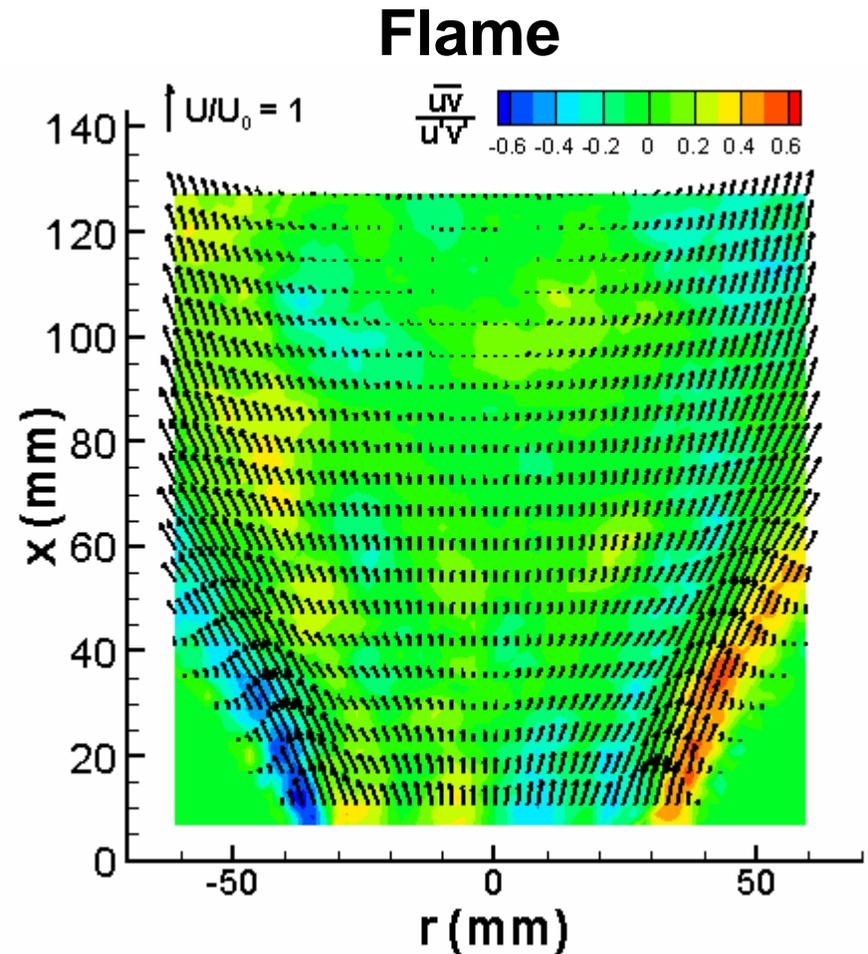
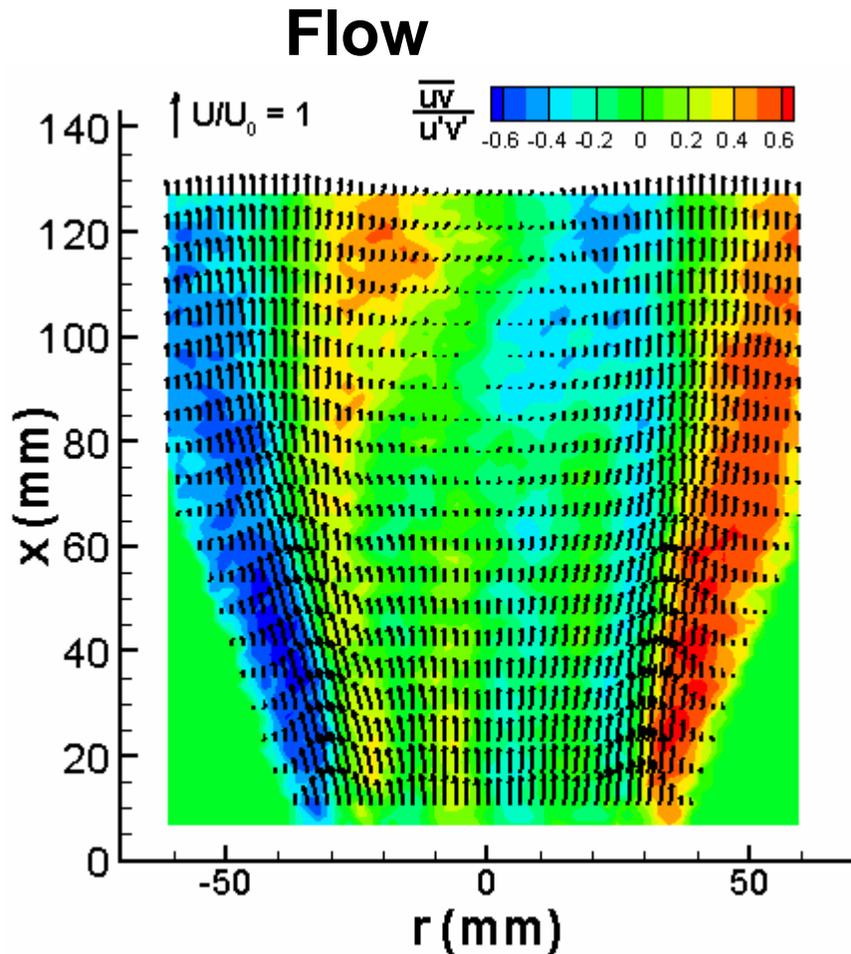
Normalized mean vectors and Reynolds stress at $U_0 = 10$ m/s



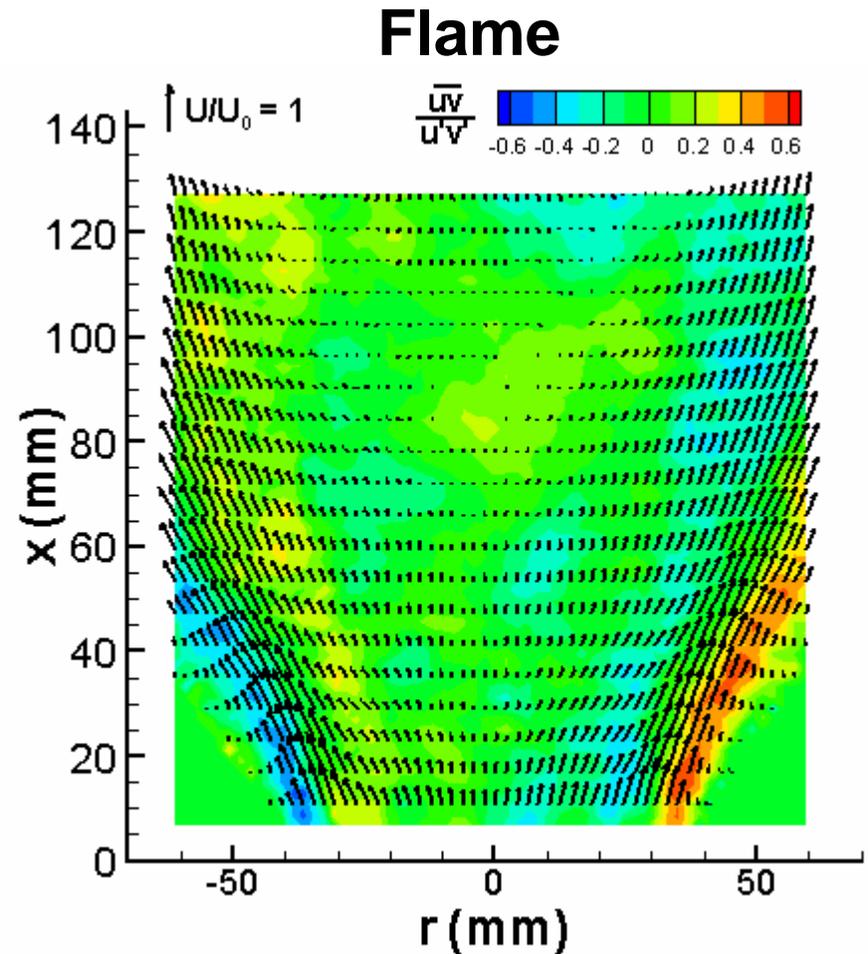
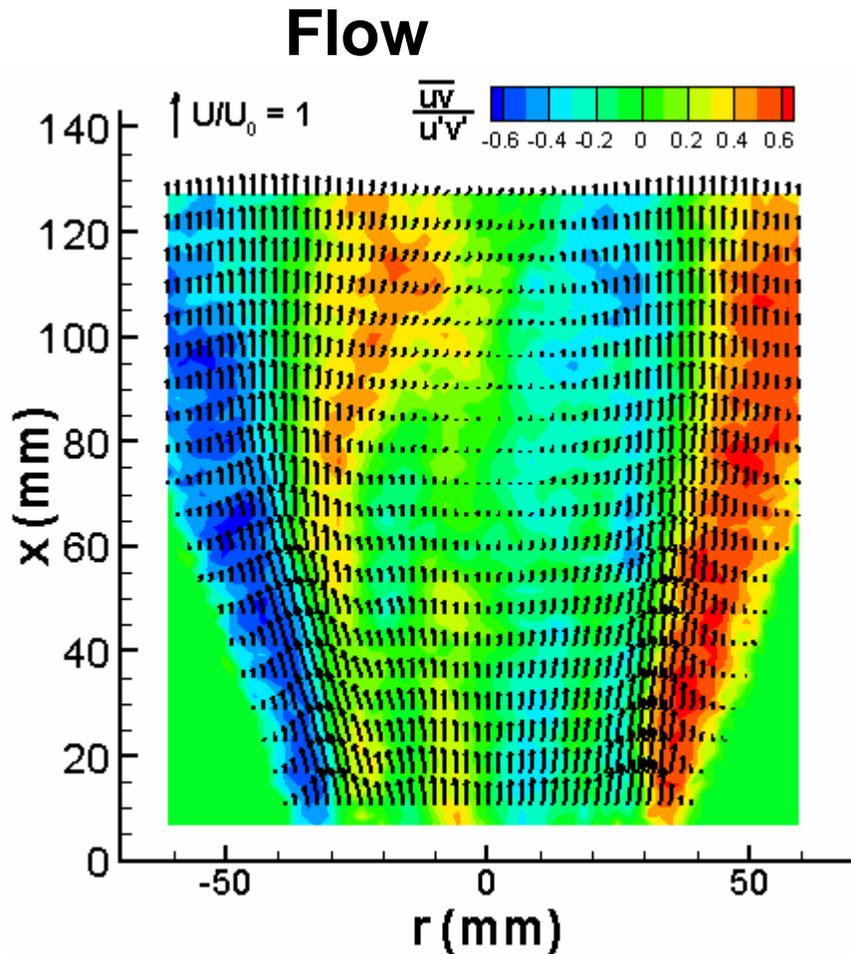
Normalized mean vectors and Reynolds stress at $U_0 = 15$ m/s



Normalized mean vectors and Reynolds stress at $U_0 = 19$ m/s

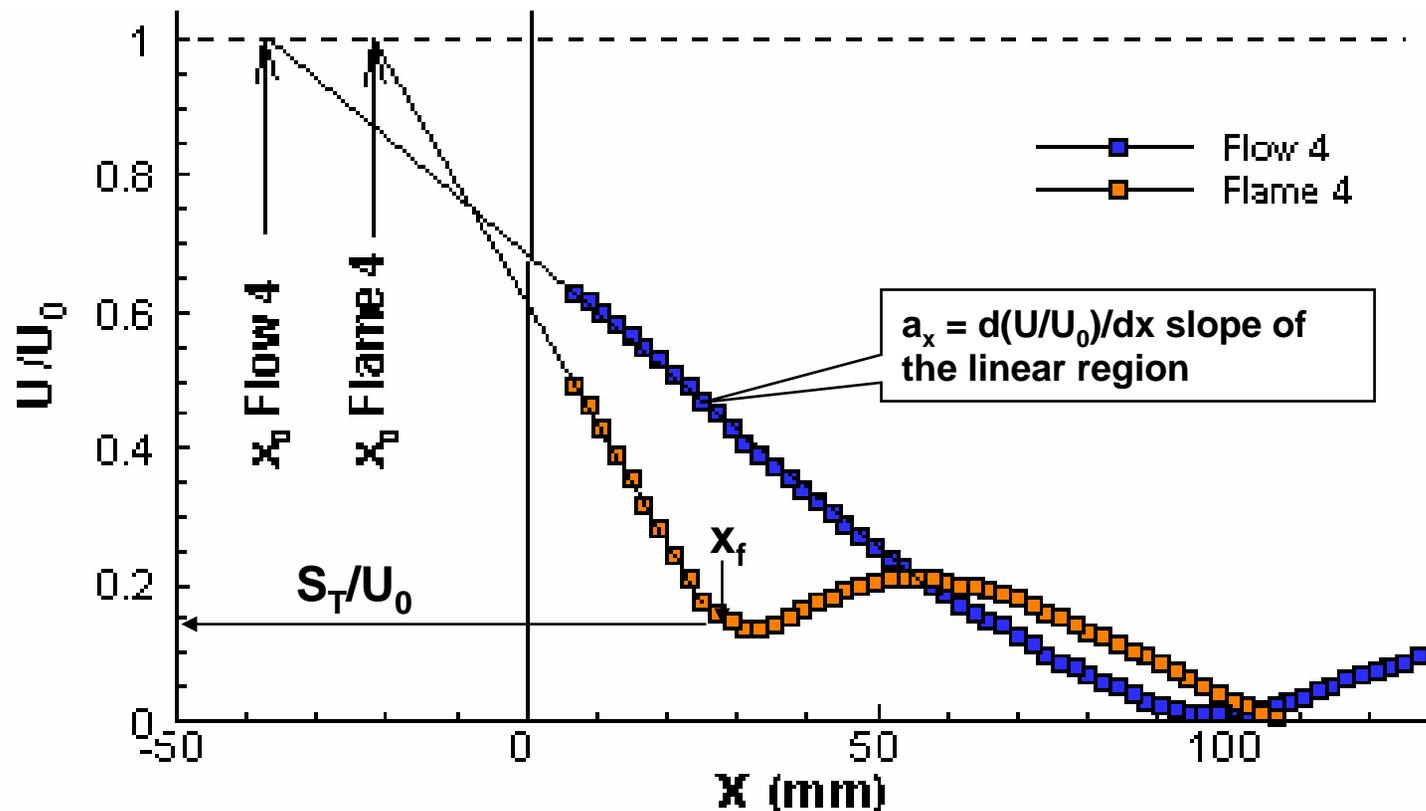


Normalized mean vectors and Reynolds stress at $U_0 = 22$ m/s

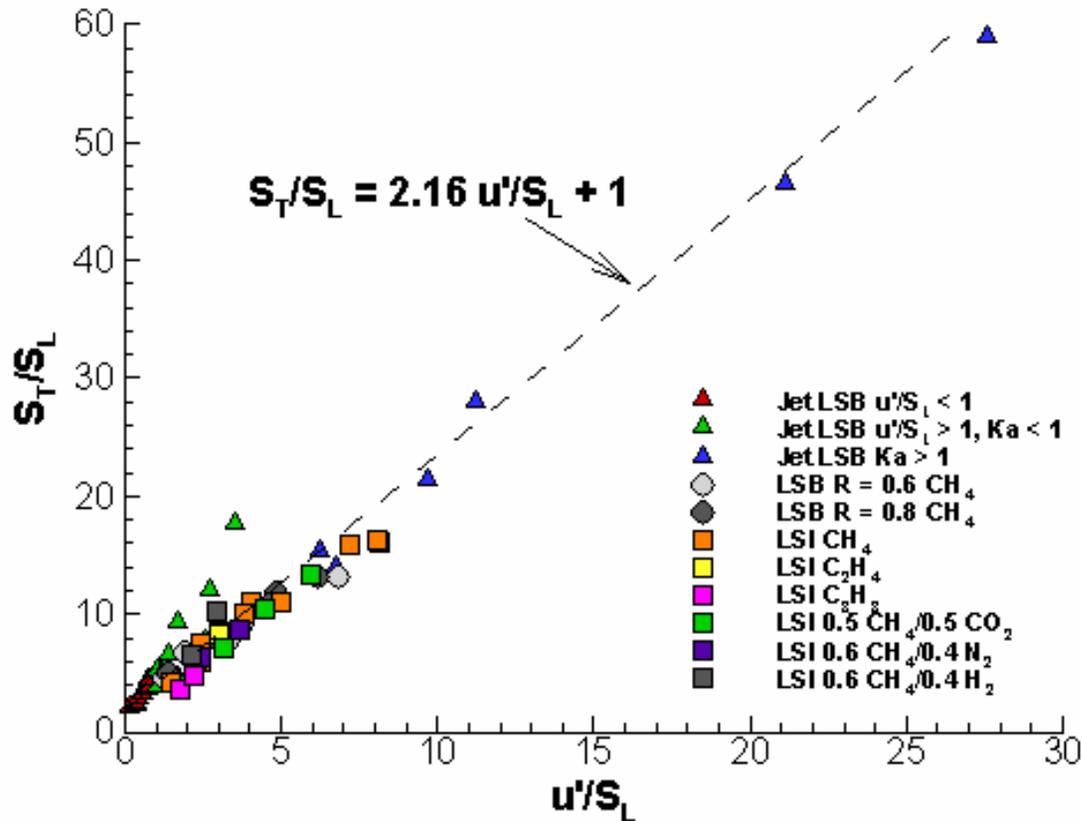


PIV Measures the Parameters that Describe Flame/Flow Coupling in LSI

- Four parameters deduced from the centerline velocity profile
Virtual Origin, x_0 , Normalized Axial Divergence Rate, a_x ,
Flame Position, x_f and Turbulent Flame Speed, S_T

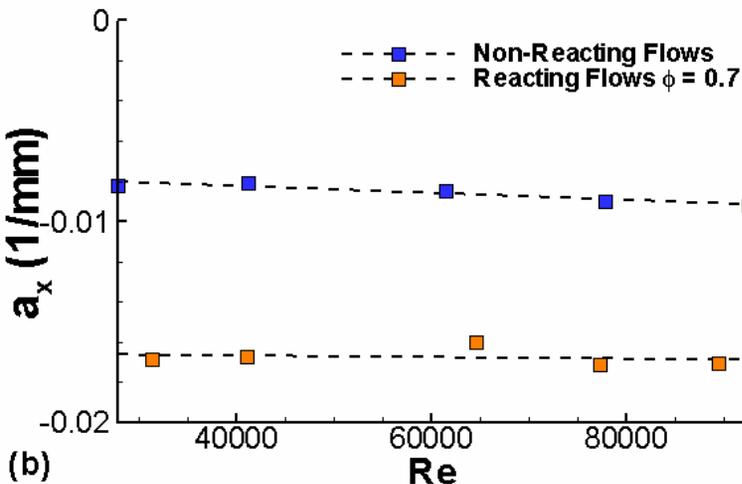
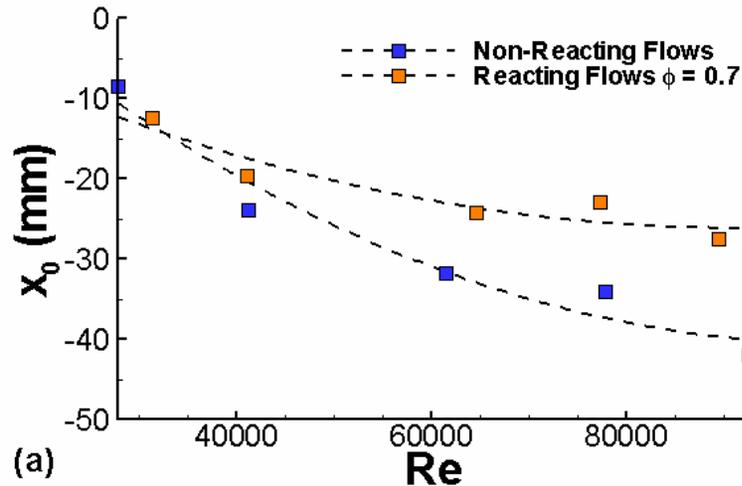


S_T of CH_4 , C_3H_8 , C_2H_4 and Diluted HC Flames Show Linear Correlation



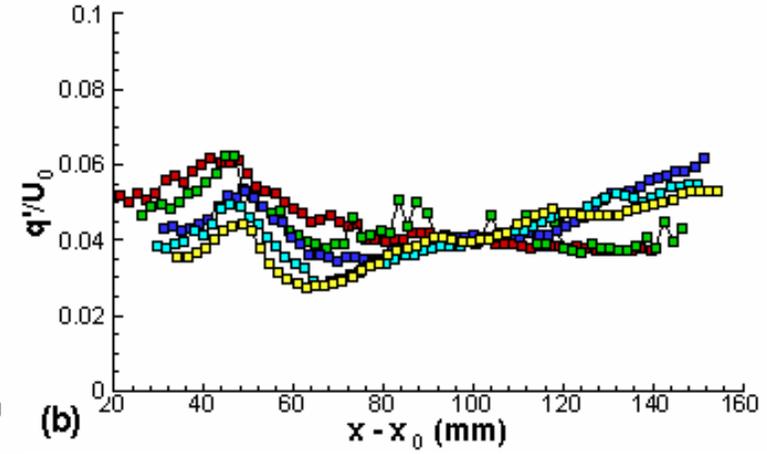
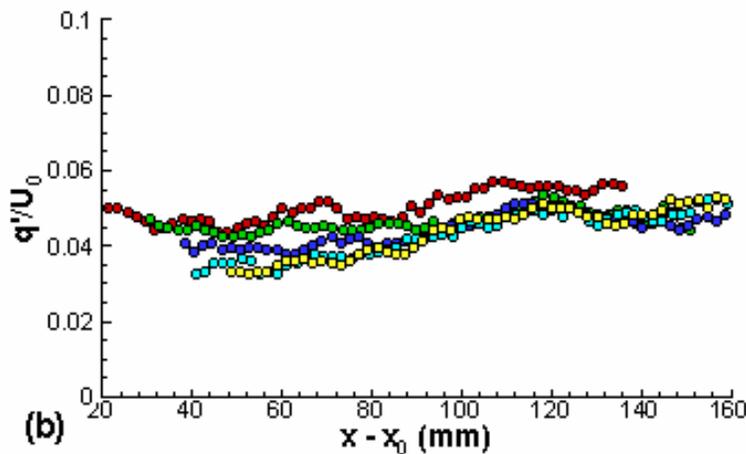
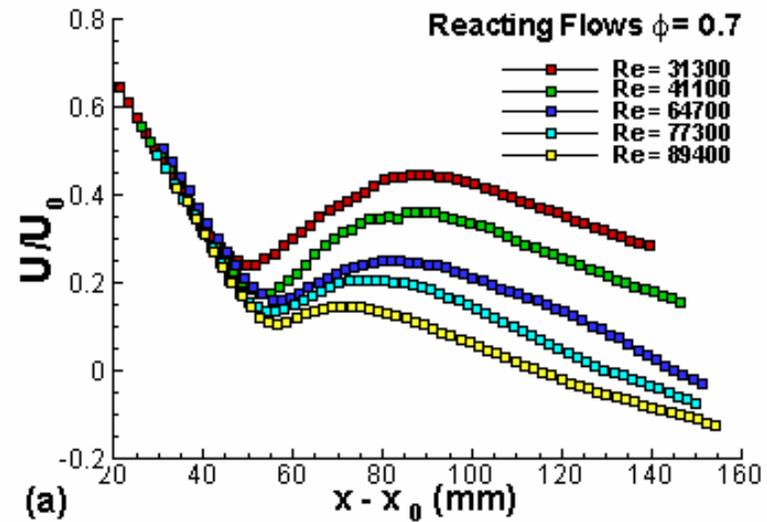
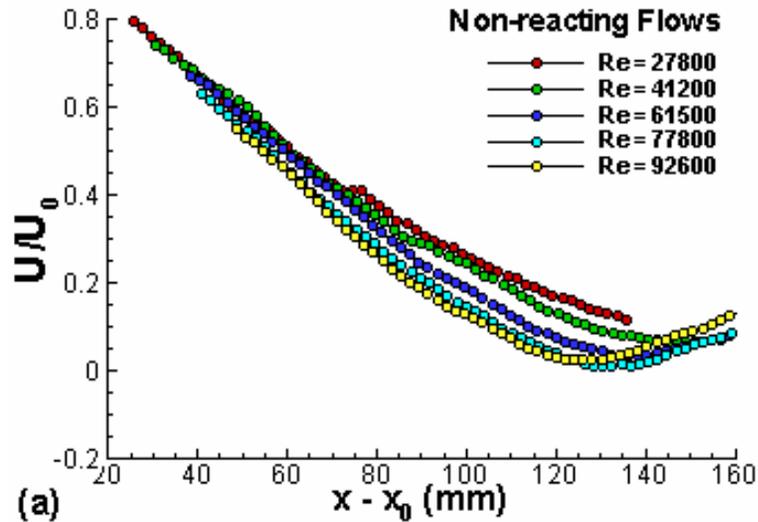
- S_T from LSI flames consistent with those of previous studies
- Linear behavior unique to low-swirl combustion
- Additional data being obtained at higher U_0

Trends of x_0 , and a_x with Reynolds Number Indicate Similarity



- **Virtual origin x_0 leveling-off at high Re**
 - ▶ Slight shift of the divergence flow structures into the injector barrel with increasing velocity
- **Normalized divergence stretch a_x insensitive to Re**
 - ▶ Combustion generates a systematic increase in a_x
- **Nearfield flow structures have a similar form that is independent of power output**

Similarity in the Nearfield Shown by Normalized Centerline Profiles



Significant Implication of Similarity

- Provides an analytical means to quantify the flame/flow relationship by the use of a_x , U_0 , S_T and x_f
 - ▶ the axial velocity at x_f is

$$U_0 - \frac{dU}{dx} (x_f - x_o) = S_T$$

- ▶ Divide through by U_0 and invoke S_T correlation gives

$$1 - \frac{dU}{dx} \frac{(x_f - x_o)}{U_0} = \frac{S_T}{U_0} = \frac{S_L}{U_0} + \frac{2.16u'}{U_0}$$

invariant due to similarity (i.e. a_x)

asymptote at large U_0

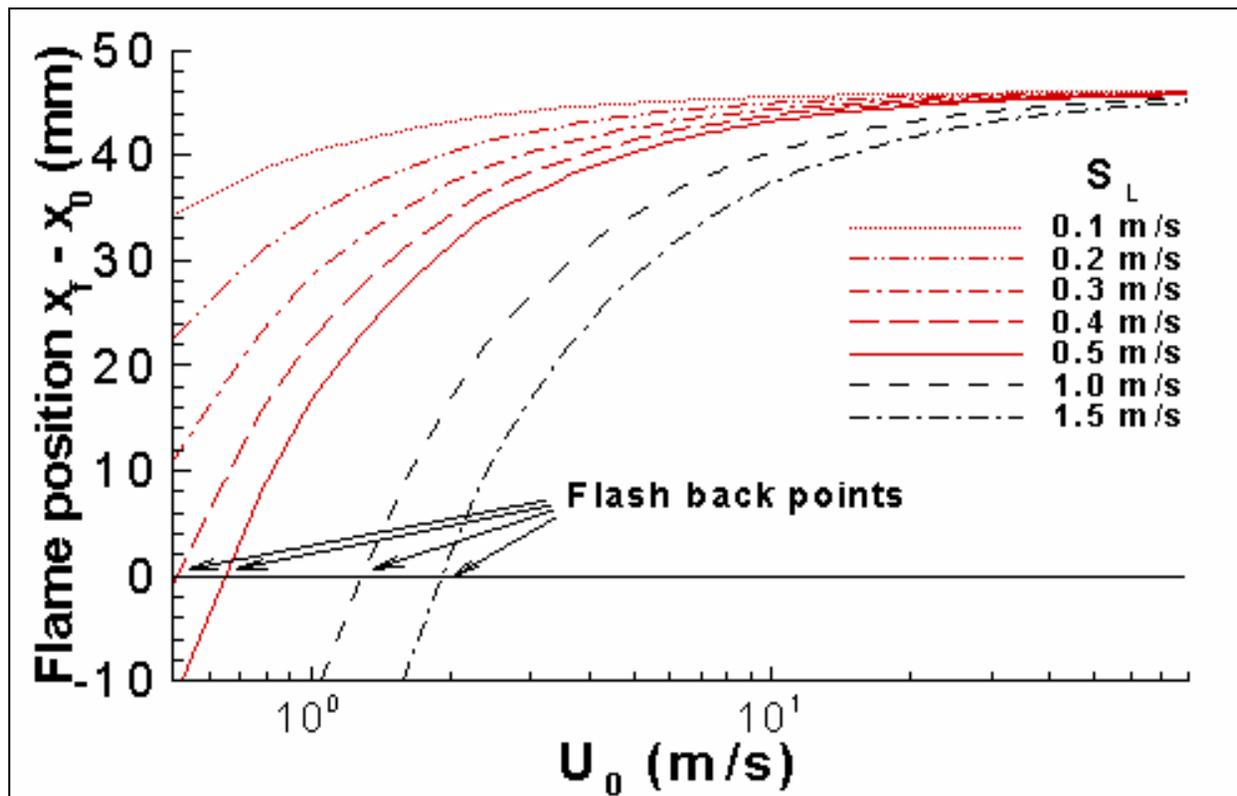
small at large U_0

constant for plate turbulence

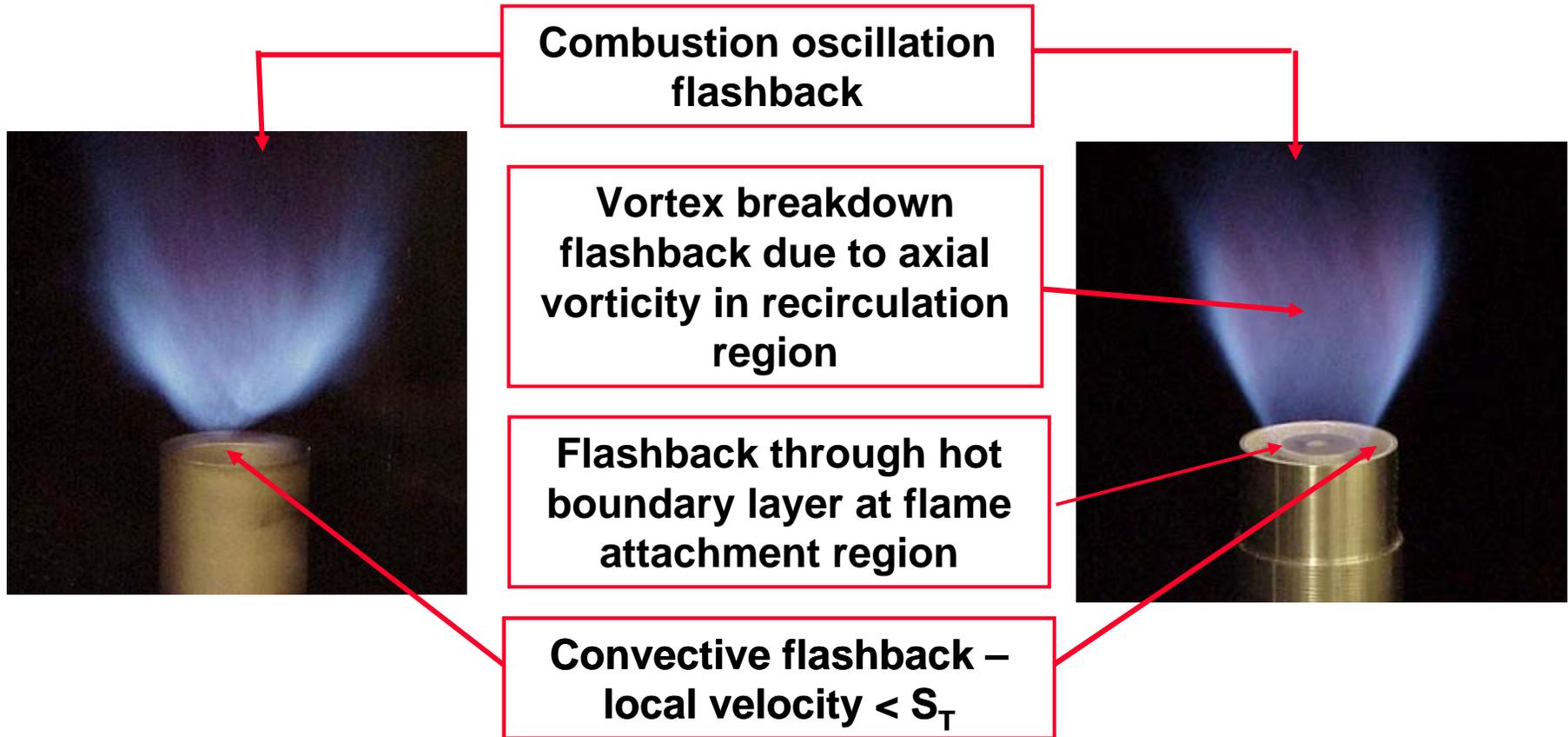
Flowfield similarity and linear S_T correlations explains why flame remains stationary through a wide range of velocities and ϕ

Flashback and Flame Positions Predictable from Analytical Equation

- Results imply that fuel effects are significant only at low U_0
 - ▶ Velocity at flash back correlates with S_L
 - ▶ Flame position independent of S_L at large U_0

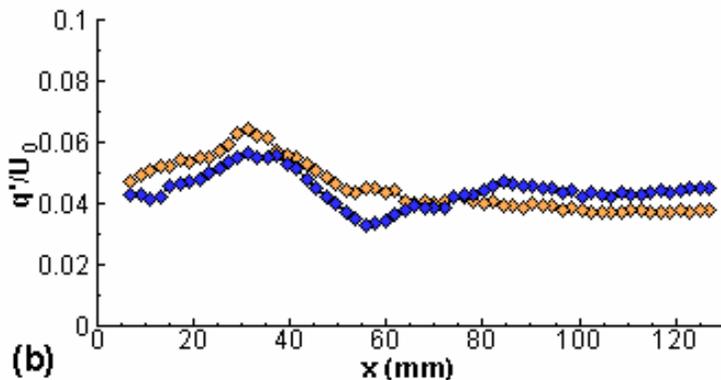
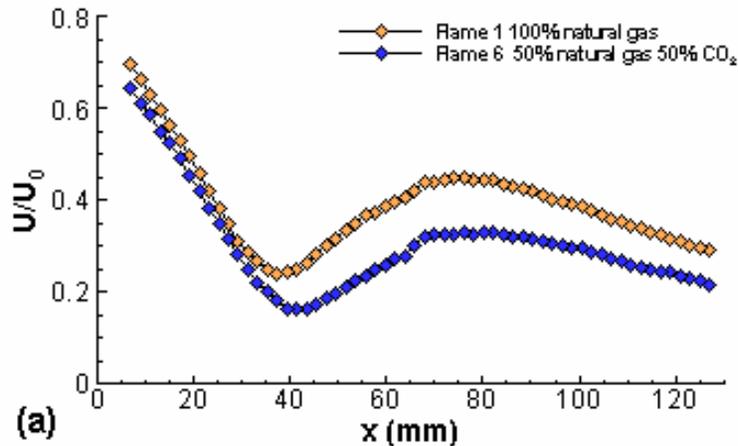


Flashback Considerations for Low-Swirl and High-swirl



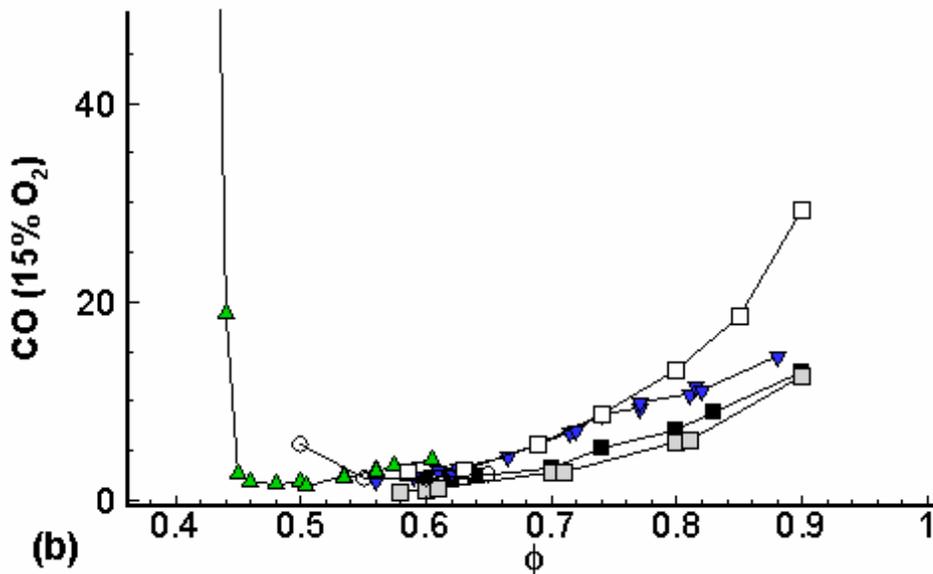
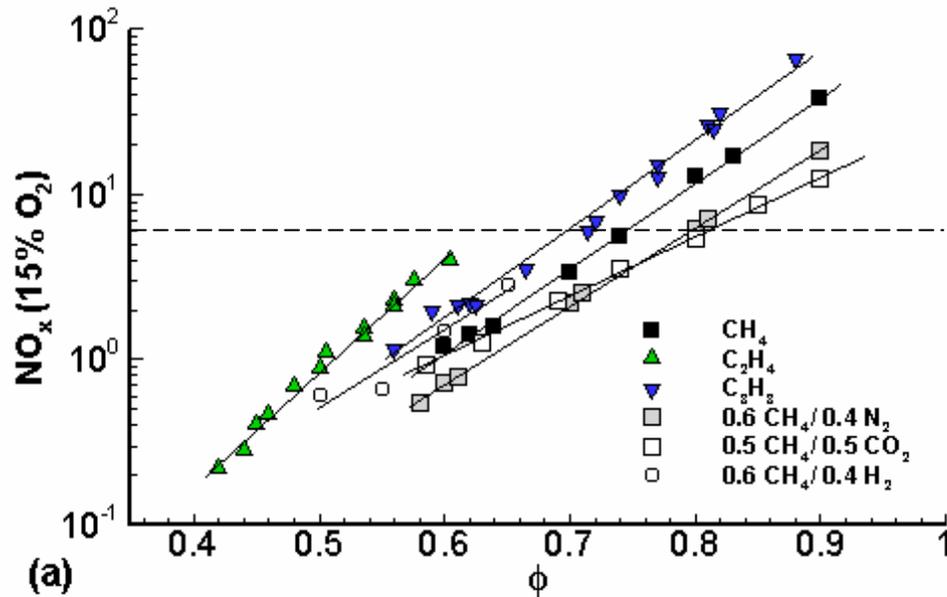
- LSC analytical model addresses convective flashback
- Need studies on LSC vulnerability to combustion oscillation flashback

Flowfield Features Unaffected by Fuel Type



- Flowfield features of CH₄ and diluted CH₄ flames are the same
- Flame stabilization mechanism not affected by variation in fuel composition
- Slight shift in flame position due to slower burning flame

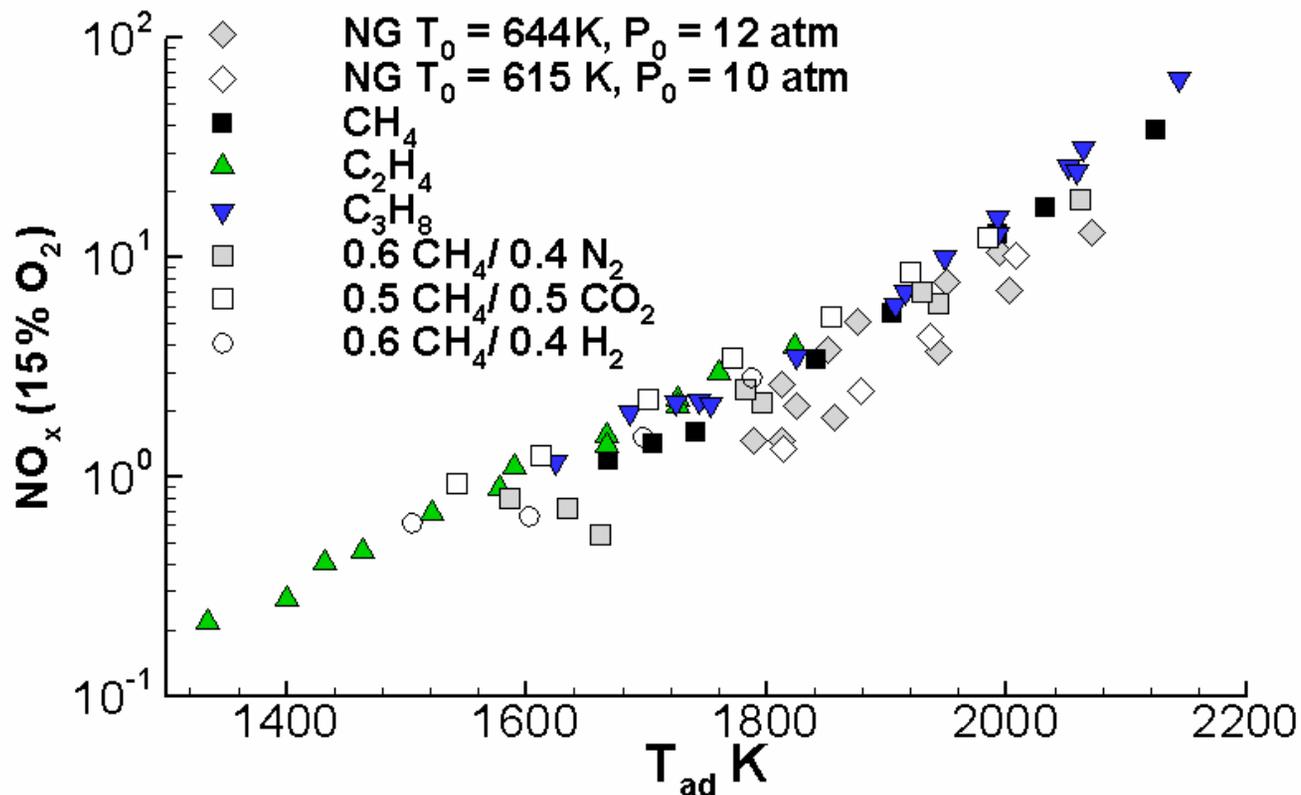
LSI Supports Stable < 5 ppm NO_x Hydrocarbon Flames



- Exponential NO_x dependence on ϕ
- CO emissions within acceptable limits

NO_x Emissions Show Log-linear Dependency on Flame Temperature

- NO_x emissions from STP laboratory experiments consistent with data at gas turbine conditions
- Absence of strong recirculation in LSI may explain the correspondence between laboratory and GT emissions



Preliminary Conclusions on Hydrocarbon Fuel-Flexibility Studies

- LSI accepts all test fuels including CH₄ diluted with H₂
- LSI supports stable < 5 ppm NO_x flames
- NO_x emissions scale with adiabatic flame temperature
 - ▶ NO_x emissions from laboratory flames STP consistent with high pressure rig test data at turbine conditions
- Significant adjustment may not be necessary for current LSI to fire with hydrocarbon fuel blends
 - ▶ S_T of hydrocarbon and CH₄ flames have same correlation
- Recent high T, P rig-tests at Solar Turbines demonstrate firing with fuels from 550 Btu/ft³ to 1250 Btu/ft³

Considerations for Adaptation of LSI to Fuel-Flexible & IGCC Turbines

- **Changes in flame speed correlation will be the 1st order effect**
 - ▶ Turbulent flame speeds for HC fuels have similar correlation as natural gas
 - Significant redesigning of swirler may not be necessary
 - ▶ **Turbulent flame speed data for H₂ mixtures are lacking**
 - **Large uncertainties in laminar flame speed data for lean H₂ mixtures**
- **Changes in heat release will be the 2nd order effect**
 - ▶ Changes in LSI flowfield correlates with combustion heat release
- **LSI swirl rate can be adjusted to accommodate the fuel effects**
 - ▶ Increase or decrease the swirl rate to optimize flame position

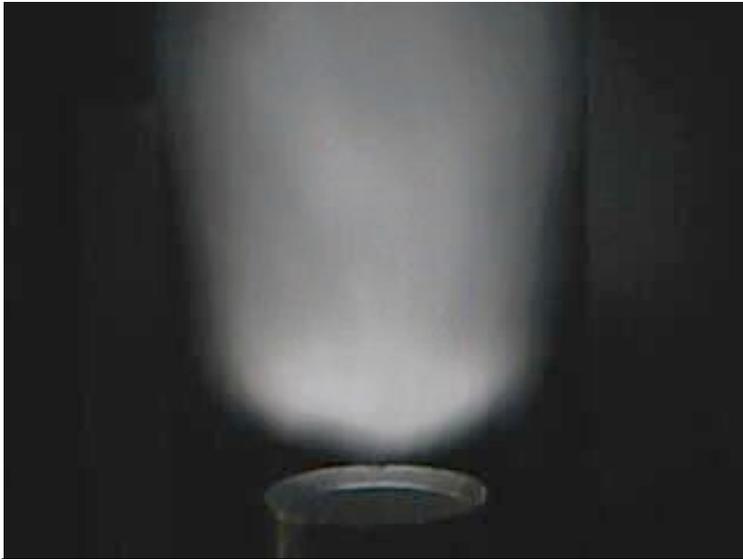
Part 4

Scale-up Issues and Challenges for IGCC

Issues and Challenges for Adapting LSI to Large Utility Turbines

- **Natural gas engines**
 - ▶ Larger sizes – increase diameter of injector
 - ▶ Larger capacity - increase of 2 to 5 times over that of LSI for Taurus 70
 - ▶ Higher operating temperatures and pressures
- **Fuel-flexible and syn-gas engines**
 - ▶ Combustion properties of syn-gas flames similar to those of low-Btu hydrocarbon fuels
 - ▶ Scaling issues + fuel injector design and premixing
- **H₂ engines**
 - ▶ Distinct characteristics of H₂ flame due to high diffusivity and reactivity
 - Premixed turbulent H₂ flames not well understood
 - Demonstration of the feasibility of LSC for H₂ flames
 - ▶ Short auto-ignition time compare to hydrocarbons
 - Engineering design for fuel injection and mixing

Laboratory Experiments on Premixed H₂ Flames in LSI



Lean premixed H₂/air flame at
 $\phi = 0.35$ & $U_0 = 20$ m/s
captured by an IR camera

- Obtained the basic knowledge to evaluate the feasibility of LSC concept for H₂
 - ▶ Measured turbulent flame speed for H₂ flames and their coupling with LSI flowfield
- Studied flames burning with pure H₂ and H₂ diluted with 25% and 50% N₂ dilution (by volume)
 - ▶ Conditions with $T_{ad} = 1200, 1320, 1380$ K
- Varied swirl number from $0.57 < S < 0.41$
 - ▶ Varied the blockage ratio and geometric pattern of the center plate
 - ▶ Much more convenient than changing swirl vane angle
- Measured lean blow-out and flowfield characteristics at $9 < U_0 < 20$ m/s
 - ▶ Define optimum LSI configuration for H₂
 - ▶ PIV data analyzed to obtain S_T correlation and other flowfield parameters

Open H₂ Flames

Attach to Burner Rim at High ϕ

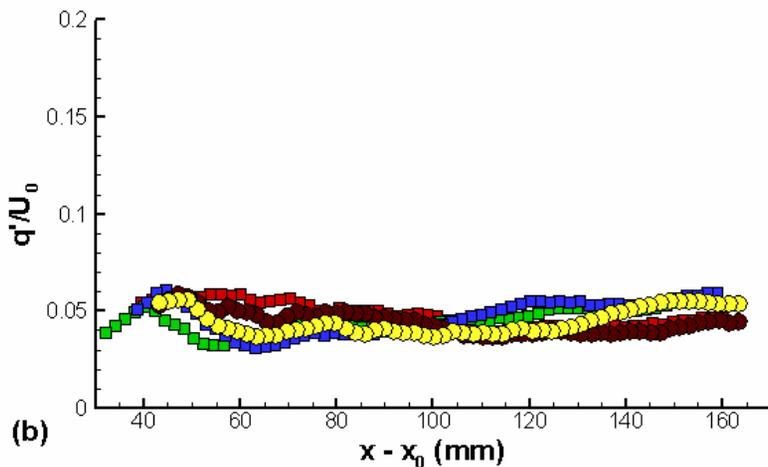
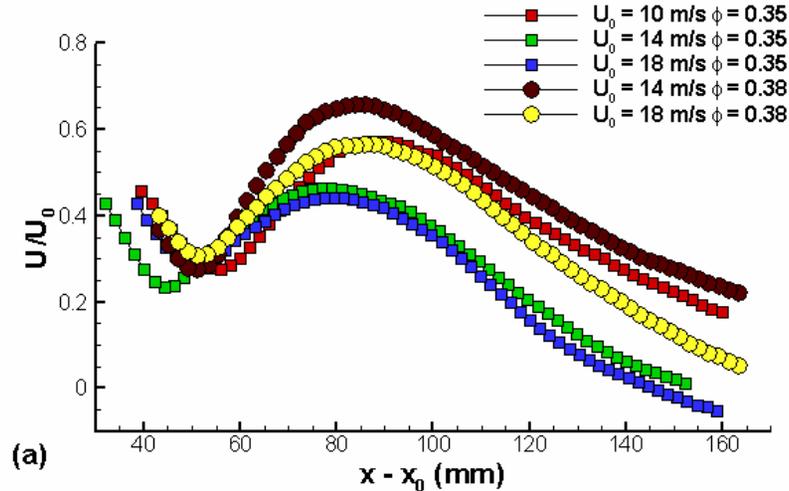


- Highly diffusive H₂ burning in low-velocity region in the shearing region between the premixture and room air
- Attachment represents an upper limit for the laboratory studies
 - ▶ Changes in the outer flowfield characteristics affect the core flow
- For turbine application, flame attachment can be mitigated by an angled expansion at the nozzle exit

Relaxing Swirl Number to Optimize for H₂ Flames

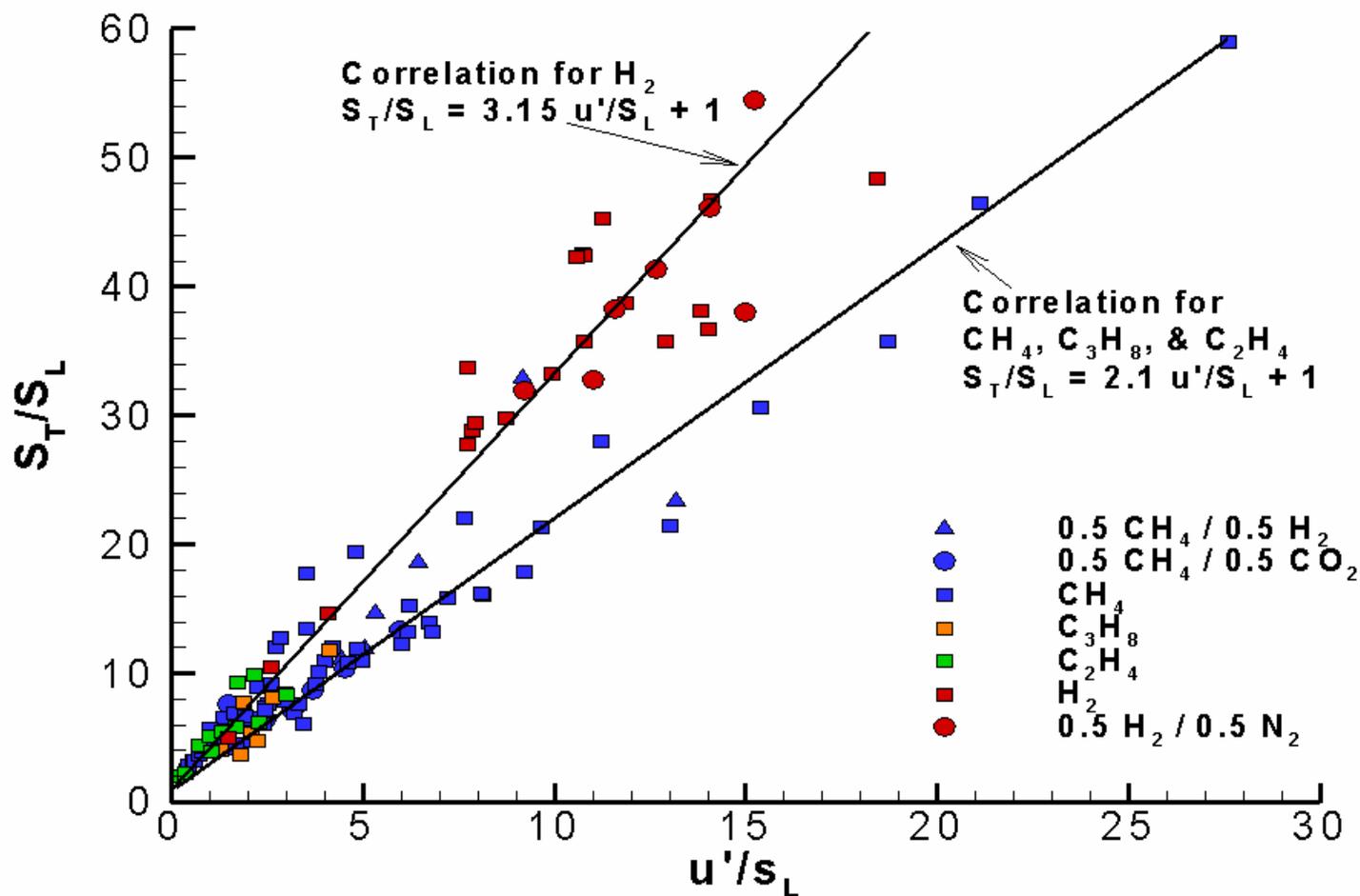
- **Current LSI (S = 0.54) stabilizes H₂ flames that are very close to the nozzle exit**
 - ▶ Flame attachment occurs for $\phi > 0.3$
- **Lowering S from 0.54 to 0.43 generates more lifted flames and pushes flame attachment to $\phi > 0.4$**
 - ▶ Flame position trend consistent with prediction of the analytical model
- **LSI with S = 0.51 offers best performance for laboratory studies**
 - ▶ LSI for H₂ is not significantly different than LSI for hydrocarbons

Flowfield Features of H₂ Flames Are Same as HC Flames



- Centerline velocity profiles show similarity features in the nearfield
- Flowfield features in the farfield are no different than those of HC flames
- Similarity is also shown by radial profiles

S_T of H_2 Flames Have Linear Correlation With a Higher Coefficient Than HC Flames



Laboratory Studies Show LSI Amenable to Burning Pure H₂

- **Dominant flame/flow coupling processes of H₂ and hydrocarbon flames are the same**
 - ▶ **Effects due to high diffusivity are impediments to open flame laboratory studies and can be addressed by engineering means**
- **Higher H₂ flame speed correlation accommodated by a small reduction of the swirl number**
- **Demonstrate the viability of the analytical model for H₂ LSI design**
- **Encouraging results useful for guiding the sub-scale H₂ LSI prototype for demonstration at high T&P**

Part 5

R&D Needs for H₂ Turbines

Many Issues and Uncertainties for Development of LSI for H₂ Turbines

- **Fundamental Issues**
 - ▶ **Ignition delay**
 - ▶ Turbulent flame speed correlation
 - ▶ Unknown properties of H₂ flames at high pressures
 - Reaction rate
 - Flame volume
 - ▶ Predict emissions at relevant conditions
 - ▶ Instability characteristics of LSI
 - ▶ Modeling & simulation
 - ▶ Flowfield/emission coupling
- **Engineering issues**
 - ▶ Scaling to larger sizes
 - ▶ Fuel injection, premixer & nozzle design
 - ▶ Verify operation at higher T & P
 - ▶ Optimize for fuel/flame properties
 - Syn-gas versus H₂
 - ▶ LSI/Combustor layout
 - ▶ Integrate with other components
 - ▶ Operations & Controls
 - H₂ flames are invisible

Developmental Steps Toward LSI for H₂ Turbines

- Applied knowledge gain from laboratory studies to optimize prototypes for H₂ (Part 4 of this talk)
- Pressurized sub-scale testing at research facilities to explore overall behavior
 - ▶ Georgia Tech. – mapping the operating regime using a small 2.54 cm diameter prototype
 - ▶ NETL – detailed investigation of scalar and velocity fields with a 5 cm diameter prototype
- Seek guidance from technical advisors on fuel compositions, firing temperatures and test conditions
- Coordinate and merge knowledge from University and Nat'l Lab researchers
- Share results with OEMs and vendors
 - ▶ Evaluate feasibility for each system
- Plan next developmental phase

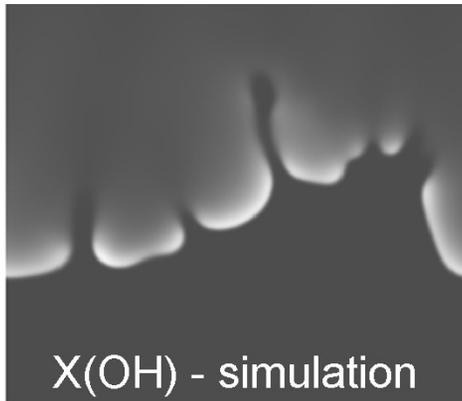
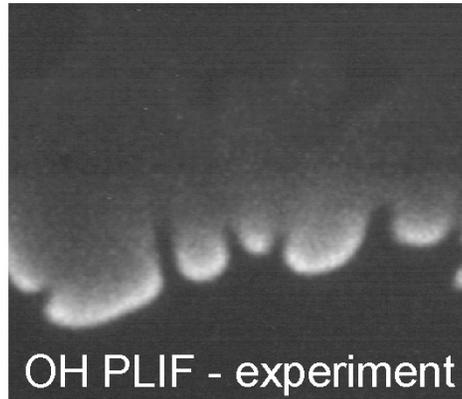
Ignition Delay, τ

- Residence time of H₂ reactants within the combustor cannot be longer than the auto-ignition delay time
 - ▶ Prevent premature ignition
- H₂ ignition delay data not yet available at relevant gas turbine temperatures and pressures
 - ▶ Measurements are being made at Univ. of California, Irvine (flow channel) and at University of Central Florida (shock tube)
- Data are encouraging for utilization of lean premixed H₂ combustion for IGCC

H₂ Turbulent Flame Speed Correlation at Relevant Gas Turbine Conditions

- **Need in-situ H₂ flame speed data for LSC development**
 - ▶ Turbulent flame speeds are strongly dependent on system configuration
 - ▶ Data measured in transient flames inside constant volume combustion vessels have non-linear characteristics that contradict the LSC results
- **Significant experimental challenges for laser anemometry measurements in high-pressure flow channels**
 - ▶ Transmitting laser and scattered signal through windows reduces signal to noise
 - ▶ Operational difficulties associated with cleanup of fine particles deposited on windows
- **University and National Lab researchers rising to this challenge**
 - ▶ **Overall trend can be inferred by observing LSI H₂ flames at high T and P**
 - ▶ Detailed measurements at selected data points for quantitative comparison with results from STP laboratory experiments

Premixed H₂ Flame Properties Not Well-Characterized at Relevant Conditions



- **Reaction zone structures of turbulent H₂ flames are different than those of hydrocarbon flames**
 - ▶ H₂ flames are inherently unstable due to preferential diffusion
 - ▶ Laboratory experiments and simulations of turbulent H₂ flame show non-uniform heat release and local extinction
 - H₂ flames may need large combustion chamber
- **Heat release model for hydrocarbon flames are not suitable for H₂ flames**
- **Research needs**
 - ▶ Improved chemical kinetics and transport for lean H₂ mixtures
 - ▶ Effects of intense turbulence and elevated temperatures and pressures on preferential diffusion effects

Predicting LSI NO_x Emissions at Relevant Conditions

- **LSI NO_x emissions correlate with flame temperature over a large range of inlet T & P**
 - ▶ Nitrogen chemistry has temperature and pressure dependencies
 - ▶ Need improved understanding of prompt NO_x versus thermal NO_x formation in ultra-lean flames
- **Laboratory experiments show impact of recirculation on NO_x formation**
 - ▶ LSI generates high NO_x when placed in a large enclosure that promotes internally recirculating flow
 - ▶ Characterizing the residence time of the hot products within the combustor may be the key to explain the differences in NO_x emissions of low-swirl and high-swirl combustion
- **Basic studies of the flowfield/emissions coupling will be beneficial to LSI development**

Combustion Dynamics of LSI

- **LSI in T70 engine shows different acoustic signature**
 - ▶ Acceptable pressure fluctuations at frequencies different than SoLoNOx high-swirl injectors
- **Pressure/heat release coupling of LSI has not been characterized**
 - ▶ Propagating nature of LSI flames and absence of well developed flow recirculation affect how the flame responds to external excitations
- **University and Nat'l Lab researchers are developing research plans to study LSI oscillation characteristics**
 - ▶ Side-by-side comparison of low-swirl and high-swirl combustion oscillation will give fresh insights to understand combustion dynamics

Summary

- **LSC is a cost-effective and robust enabling ultra-low emissions technology for natural gas turbines and shows very good promise for fuel-flexible turbines utilizing blended hydrocarbon fuels**
- **Principle of LSC is described by top level model that can be used to guide its scale-up for large utility turbines**
- **LSC shows good promise for large turbines operating on natural gas and other hydrocarbons**
- **Development for syngas and H₂ turbines requires empirical inputs as well as insights on fundamental combustion properties of H₂ flames to guide the adaptation**

Thank You for Your Attention

We Want Your Feedback

- If you are in the audience please send an e-mail to ronwolk@aol.com acknowledging your participation in this web-cast
- For questions and comment on the technology, please e-mail me RKCheng@lbl.gov